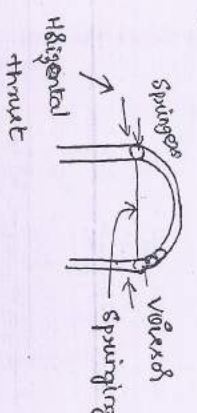


\* Individual shape of plate used for fabrication of an arch is called veissels.

\* Lower most veissels are called springers.

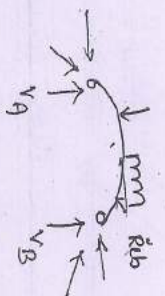
rib-bend steel bars



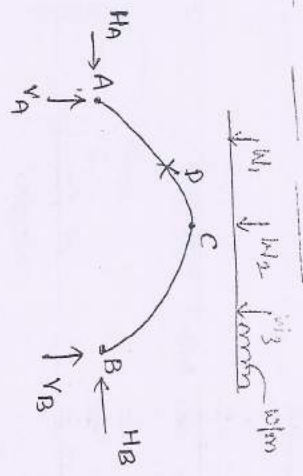
\* Force developed along the tangent is called Normal thrust.

\* Force developed  $1^\circ (90^\circ)$  to the Normal thrust is called Radial shear.

\* Top most curve point on the arch is called Crown.



FOR STATIC LOADS:-



Consider a 3-hinged arch subjected to loads shown in figure. Since ends are hinged there will be reaction components at each namely horizontal & vertical. Here there are ~~total~~ four components namely,  $H_A, V_A$  &  $H_B, V_B$

In any plane,  $\sum F_H = 0, \sum F_V = 0, \sum M_A \neq M_B = 0$ .

In this case, thrust is solved by  $M_C = 0$ , since it is hinged.

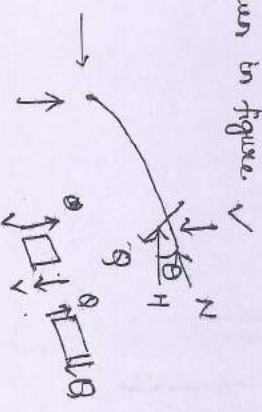
$\therefore \sum F_H = 0, \Rightarrow H_A = H_B$ , say H

Since, the loads tend to spread the arch, the horizontal

thrust is in the inward direction as shown in figure. Consider a section at D,

V, be the Vertical Shear  
Q, be the Radial Shear  
N, Normal thrust

all these forces are shown in their positive sense as shown. Let the normal to the section make an angle  $\theta$  with



Horizontal

$N = V \sin \theta + H \cos \theta$

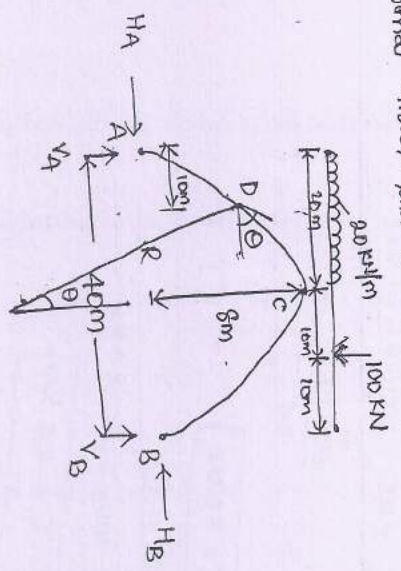
Then,

$Q = V \cos \theta - H \sin \theta$

The normal stress can be obtained by considering forces including the reaction on any one part of arch.



A 3-hinged arch hinged at the spring and crown points has a span of 40m and a central rise of 8m. It carries a UDL of 20 kN/m over the left half of the span together with a concentrated load of 100 kN at right quarter span point. Find the reactions at the supports, normal thrust and shear at a section 10m from left support.



Given,

$L = 40\text{ m}$

$h = 8\text{ m}$

$V_A, V_B, H_A, H_B = ?$

$N = ?$

$Q = ?$

1. REACTIONS:-

$\downarrow \sum F_V = 0$

$V_A + V_B = 20 \times 20 + 100$

$V_A + V_B = 500 \text{ --- (1)}$

ii,  $\sum M_B = 0$ , gives

$V_A \times 40 - 20 \times 20 \left( \frac{20+20}{2} \right) - 100 \times 10 = 0$

$40 V_A = 20 \times 20 \times 30 + 1000$

$V_A = \frac{13000}{40} = 325 \text{ kN}$

we get  $V_B = 500 - 325 = 175 \text{ kN}$

$\therefore V_A = 325 \text{ kN}$  &  $V_B = 175 \text{ kN}$

iii, since C is hinged,  $M_C = 0$  gives

$V_B \times 20 - 100 \times 10 - H \times 8 = 0$

$175 \times 20 - 100 \times 10 = 8H$

$H = 312.5 \text{ kN}$

$\therefore H_A = H_B = H = 312.5 \text{ kN}$

2. finding the value of R & S

Let D be the point 10m from left support where the normal thrust & shear are to found. Now, from property of

$\frac{L}{2} \times \frac{L}{2} = h(2R-h)$   $R = \frac{L^2}{8h} + \frac{h}{2}$

$\frac{40}{2} \times \frac{40}{2} = 8(2R-8)$

$\frac{1600}{4} = 16R - 64$

$\frac{1600 + 64}{4} = 16R$

$\frac{1664}{4} = 16R$   $\frac{1600 + 8 \times 8}{4} = \frac{1600 + 64}{4}$

$R = \frac{1664}{16} = 104$   $= \frac{1600 + 64}{64} = \frac{1664}{64} = 26$

$R = \frac{1664}{16} = 104$   $\therefore R = 29 \text{ m}$

$\therefore$  slope at  $D = \theta = \sin^{-1} \left( \frac{R}{L} \right) = \sin^{-1} \left( \frac{10}{29} \right) = 20$

Answer V at 12.5m:-

$$\therefore V = V_A - 20 \times 10 = 98.5 - 200 = 12.5 \text{ KN}$$

$$\therefore V = 12.5 \text{ KN}$$

thrust,  $N = V \sin \theta + H \cos \theta$   
 $= 12.5 \sin 29.191^\circ + 312.5 \cos 29.191^\circ$

$$N = 336.437 \text{ KN}$$

at shear,  $R = V \cos \theta - H \sin \theta$   
 $= 12.5 \cos 29.191^\circ - 312.5 \sin 29.191^\circ$

$$\therefore R = 9.575 \text{ KN}$$

travels each to span 25m with central raise 5m is hinged at the pin and springing. It carries a point load of 100 kN at 6m from the left support. Calculate reactions at supports, reactions at crown and moment at 5m from the left support.

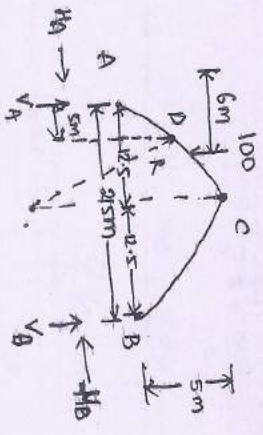
$$L = 25 \text{ m}$$

$$h = 5 \text{ m}$$

Q1)  $V_A, V_B, H_A, H_B = ?$

$$H_C, V_C = ?$$

$$M_x \text{ at } x = 5 = ?$$



$$\downarrow \sum F_y = 0$$

$$V_A + V_B = 100 \text{ KN} \text{--- (1)}$$

$$\uparrow \sum M_B = 0,$$

$$V_A \times 25 - 100(25-6) = 0$$

$$V_A = \frac{100 \times 19}{25} = 76 \text{ KN}$$

$$\therefore V_A = 76 \text{ KN}$$

Sub. in (1)

$$V_B = 100 - 76 = 24 \text{ KN}$$

$$\therefore V_B = 24 \text{ KN}$$

$$\text{iii) } \sum M_C = 0, \text{ since C is hinged}$$

~~0~~  $\times$  ~~0~~ !

$$-V_B \times 12.5 + H \times 5 = 0$$

$$5H = 24 \times 12.5$$

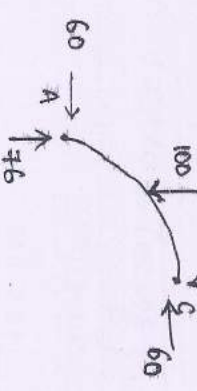
$$\therefore H = 60 \text{ KN}$$

$$\therefore H_A = H_B = 60 \text{ KN}$$

2) REACTIONS AT CROWN:-

Considering equilibrium of the left half of the arch, the

reaction at crown are



$$\therefore H = 60 \text{ KN}$$

$$V_C = 24 \text{ KN}$$



From property of circle,

$$i, \quad \therefore R = \frac{L^2}{8h} + \frac{h}{2}$$

$$= \frac{25^2}{8 \times 5} + \frac{5}{2}$$

$$R = 18.125 \text{ m}$$

$$ii, \quad \alpha = \frac{L}{2} - R \sin \theta$$

$$5 = \frac{25}{2} - R \sin \theta$$

$$\therefore R \sin \theta = 12.5 - 5 = 7.5$$

$$\sin \theta = \frac{7.5}{R} = \frac{7.5}{18.125} = 0.4138$$

$$\theta = 24.443^\circ$$

$$iii, \quad y_D = h - R(1 - \cos \theta)$$

$$= 5 - 18.125(1 - \cos 24.443^\circ)$$

$$y_D = 3.375$$

$$\therefore M_D = V_A \times 5 - H \times y_D$$

$$= 76 \times 5 - 60 \times 3.375$$

$$M_D = 177.5 \text{ kNm}$$

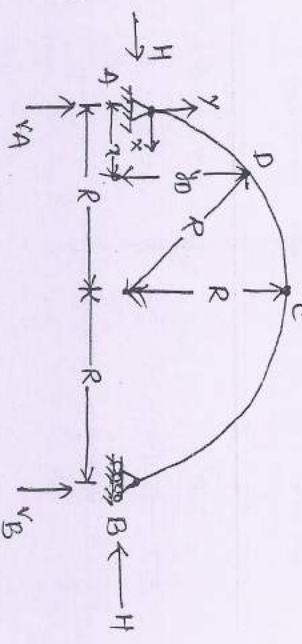
intensity w/joint length same as if it were a horizontal beam. Determine the reactions, supports and maximum bending moment in the arch.

Given: It's semi-circular arch,  $L=2R$ .

$$h=R$$

$$L=2R$$

$V_A, V_B$ ?  
 $H_A, H_B$ ?  
 $M_{ax}, M_{min}$ ?



1. Reaction: Due to symmetry

$$V_A = V_B = \frac{w \times 2R}{2} = wR$$

$$\therefore V_A = V_B = wR$$

Taking moment about C, we get  $\therefore M_C = 0$

$$H \times R - V_A \times R + wR \times \frac{R}{2} = 0$$

$$H \times R - wR \times R + \frac{wR^2}{2} = 0$$

$$R \times H = wR^2 - \frac{wR^2}{2}$$

$$\therefore H = \frac{wR - wR}{2}$$

$$\therefore H = \frac{wR}{2}$$

2. Bending moment:-

$$i, \quad \alpha = \frac{L}{2} - R \sin \theta$$

$$ii, \quad y = h - R(1 - \cos \theta)$$

$$\therefore \alpha = R - R \sin \theta$$

$$\therefore y = R \cos \theta$$

$$M_x = V_A \times x - H_A y - W \times x \times \frac{x}{2}$$

$$= WR(R - R \sin \theta) - \frac{WR}{2} \times R \cos \theta - \frac{W}{2} (R - R \sin \theta)^2$$

$$= \frac{WR^2}{2} [2(1 - \sin \theta) - \cos \theta - (1 - \sin \theta)^2]$$

$$= \frac{WR^2}{2} [2 - 2 \sin \theta - \cos \theta - 1 + 2 \sin \theta - \sin^2 \theta]$$

$$M_x = \frac{WR^2}{2} (1 - \cos \theta - \sin^2 \theta) //$$

For  $M_x$  to be maximum,

$$\frac{dM_x}{d\theta} = 0,$$

$$\Rightarrow \frac{WR^2}{2} (+ \sin \theta - 2 \sin \theta \cos \theta) = 0$$

$$\therefore \sin \theta (1 - 2 \cos \theta) = 0$$

$\theta = 0$  gives the crown point where moment is zero (minimum)

$1 - 2 \cos \theta = 0$  should give maximum point

i.e.  $\cos \theta = 0.5$

$$\Rightarrow \theta = 60^\circ$$

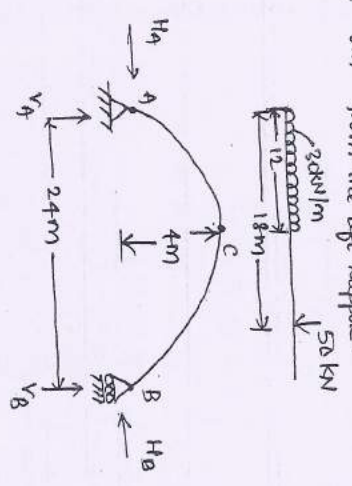
$\therefore$  though  $\theta$  is maximum point,  $\pi = R(1 - \cos 60^\circ)$

$$\therefore M_{\max} = \frac{WR^2}{2} (1 - \cos 60^\circ - \sin^2 60^\circ)$$

$$= \frac{WR^2}{2} \left(1 - \frac{1}{2} - \frac{3}{4}\right) = \frac{WR^2}{2} \left(\frac{2-2-3}{4}\right)$$

$$M_{\max} = -\frac{WR^2}{8}$$

has a span of 24m and a central rise of 4m. It carries a concentrated load of 50 kN at 18m from left support and a uniformly distributed load of 30 kN/m over the left-half portion. Determine the moment, thrust and radial reaction at a section 6m from the left support.



Given,  
 $L = 24m$   
 $h = 4m$

$V_A, H_A, V_B, H_B, N, Q = ?$

1. Reactions at supports:-

$$\sum F_y = 0$$

$$V_A + V_B = 50 + 30 \times 12$$

$$\therefore V_A + V_B = 410 \text{ kN} \quad \text{--- (1)}$$

$$\sum M_B = 0,$$

$$V_A \times 24 - 50 \times 6 - 30 \times 12 \left(\frac{12}{2} + 12\right) = 0$$

$$V_A = 282.50 \text{ kN}$$

Sub.  $V_A$  in (1), we get

$$V_B = 410 - 282.50 = 127.5 \text{ kN}$$

$$\therefore V_B = 127.5 \text{ kN}$$



$\sum M_c = 0,$

$V_B \times 12 - H \times 4 - 50 \times 6 = 0$

$127.5 \times 12 - H \times 4 - 50 \times 6 = 0$

$\therefore H = 307.5 \text{ KN}$

2. Moment At 6m from left supports

$M = V_A \times 6 - H \times y_D - 30 \times \frac{6^2}{2}$

$\therefore$  In the parabolic arch,

$y = \frac{4hx(L-x)^2}{L^2}$

$\therefore$  at  $x=6,$   
 $y_D = \frac{4 \times 4 \times 6 \times (24-6)^2}{24^2} = 3\text{m}$

$\therefore M = 282.5 \times 6 - 307.5 \times 3 - 30 \times \frac{6^2}{2}$

$\therefore M = 232.5 \text{ kNm}$

3. Vertical shear at D,

$V = V_A - 30 \times 6$   
 $= 282.5 - 30 \times 6 = 102.5 \text{ KN}$

$\therefore V = 102.5 \text{ KN}$

4. slope is given by  $y = \frac{4hx(L-x)^2}{L^2}$

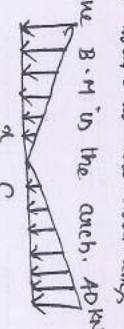
$\therefore \frac{dy}{dx} = \tan \theta = \frac{4h(L-2x)}{L}$

$\tan \theta = \frac{24 \times 4 - 2 \times 5}{24 \times 24}$

$\theta = 18.435^\circ$

$N = V \sin \theta + H \cos \theta = 324.133 \text{ KN}$   
 $\oplus = V \cos \theta - H \sin \theta = 0$

\* A symmetric three-hinged parabolic arch has a span of 30m and of 6m. The arch carries a distributed load which varies uniformly at each abutment to zero at mid-span. determine  
 (a) The horizontal thrust at the abutments  
 (b) maximum thrust B.M in the arch. 40 marks



Due to symmetry

$V_A = V_B = \frac{1}{2} \times \text{total load}$   
 $= \frac{1}{2} \times (\frac{1}{2} \times 15 \times 40) = 300 \text{ KN}$

$\sum M_c = 0,$  gives

$V_A \times 15 - H \times 6 - \frac{1}{2} \times 15 \times 40 \times \frac{2}{3} (15) = 0$   
 $H = 850 \text{ KN}$

Taking origin at crown point C, the equation of parabola is given by

$\frac{x^2}{a} = \text{slope} = \alpha$

at  $x=15, y=6,$   
 $\therefore \alpha = \frac{15^2}{6} = 37.5$   
 $\therefore x^2 = 37.5 \times y$  or  $y = \frac{x^2}{37.5}$

$$= 250 \times \frac{x^2}{3.75} - \frac{1}{2} \times x \times \frac{x}{15} \times 40 \times \frac{x}{3}$$

$$= \frac{250x^2}{3.75} - \frac{40x^3}{45}$$

$$\therefore M_{2 \max}, \frac{dM_1}{dx} = 0 \Rightarrow \frac{250}{3.75} 2x - \frac{40}{45} 3x^2 = 0$$

$$\therefore x \left[ \frac{800}{3.75} - \frac{40}{15} x \right] = 0$$

$$\therefore x = 0 \text{ and } \frac{800}{3.75} - \frac{40}{15} x = 0$$

$\therefore x = 0$ , gives mean point, where B.M is zero. Here, the point of maximum

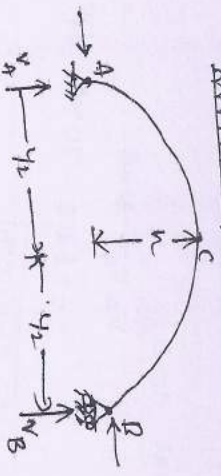
moment is given by  $x = 11$

$$x = \frac{15}{40} \times \frac{400}{3.75} = 5$$

$$\therefore M_{\max} = \left[ \frac{250x^2}{3.75} \right] - \left[ \frac{40x^3}{45} \right]$$

$$= 55.555 \text{ kNm}$$

\* Show that the parabolic shape is funicular shape for a 3-binged arch subjected to U.D.L over its entire span.



Let the span of the arch be L and together as shown in figure due to symmetry,

$$V_A = V_B = \frac{1}{2} \times \text{total load}$$

$$\therefore V_A = V_B = \frac{1}{2} \times w \times L = \frac{wL}{2}$$





$$V_A \times \frac{L}{2} - H \times h - W \times \frac{L}{2} \times \frac{L}{4} = 20$$

$$WL \times \frac{L}{2} - H \times h - \frac{WL^2}{8} = 0$$

$$H \times h = \frac{WL^2}{8} - \frac{WL}{8}$$

$$\boxed{H = \frac{WL^2}{8h} - \frac{WL}{8}}$$

at any section distance  $x$  from A,

$$M = V_A \times x - H \times y - \frac{WLx}{2}$$

$$\therefore \text{Put in a parabolic arch, } y = \frac{4hx(L-x)}{L^2}$$

$$M = \frac{WL}{2}x - \frac{WLx}{8} \cdot \frac{4hx(L-x)}{L^2} - \frac{WLx}{2} = 0$$

$$= \frac{WLx}{2} - \frac{W}{2} \left( \frac{4x(L-x)}{L} \right) - \frac{WLx}{2} = 0$$

Thus, for a parabolic arch subjected to a UDL over its entire span, the B.M at any section is zero. Hence, the parabolic shape is a funicular shape.

For a 3-hinged arch subjected to UDL over entire span.

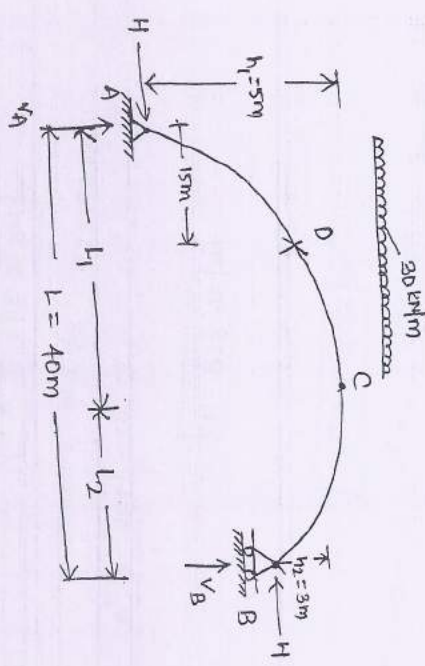
$$\frac{WL^2}{2} - H \times h = 20$$

$$H \times h = 20$$

$$H = \frac{20}{h}$$

$$H = \frac{20}{3} = 6.67$$

Figure shows a UDL of intensity 30 kN/m over the portion BC of the horizontal thrust developed. Determine the horizontal thrust developed. Also the B.M, normal thrust and shear force developed at a section 15m from the left support.



Taking C as the origin, the eqn of parabola is

$$\frac{y^2}{a^2} = a - x, \text{ where } a \text{ is a constant}$$

Let the horizontal distance b/w A and C be  $L_1$  and that of C and B be  $L_2$ .

$$\frac{L_1^2}{a^2} = a - \frac{L_1}{3}$$

$$\therefore \frac{L_1^2}{a^2} = \frac{L_1 + L_2}{3} = \frac{L}{3}$$

$$\therefore L_1 = \frac{LV\sqrt{3}}{V_3 + V_3} = \frac{40\sqrt{3}}{V_3 + V_3} = 20.54 \text{ m}$$

$$\therefore L_2 = 40 - 20.54 = 19.46 \text{ m}$$

$\sum M_C = 0$ , gives

$$V_B \times 19.46 = H \times 3$$

$$H = \frac{19.46}{3} V_B$$

$$\therefore H = 5.82 V_B \quad \text{--- (1)}$$

$$V_B \times 40 + 4 \times 2 = 30 \times 22.54 \times \frac{22.54}{2}$$

$$= 7620.774$$

from (1)

$$\therefore V_B = 147.58 \text{ kN}$$

$$H = 5.82 \times 147.58 = 858.92$$

$$\therefore H = 858.92 \text{ kN}$$

and  $V_A = 30L_1 - V_B$

$$= 30 \times 22.5 - 147.58 = 528.02 \text{ kN}$$

$$\therefore V_A = 528.02 \text{ kN}$$

The portion left of C may be treated as a parabola of span

$$L' = 2 \times L_1 = 2 \times 22.54 = 45.08 \text{ m}$$

unlike, equation of parabola is

$$y = \frac{4h_1 x (L' - x)}{L'^2} = \frac{4(5)(x)(45.08 - x)}{45.08^2}$$

$$x = 15 \text{ m}, y = 4.44 \text{ m}$$

Therefore,  $\eta$  at this section

$$\eta = V_A \times 15 - H \times 4.44 - 30 \times 15 \times \frac{15}{2}$$

$$= 528.02 \times 15 - 858.92 \times 4.44 - 30 \times 15 \times \frac{15}{2}$$

$$= 340 \text{ kNm}$$

$$\frac{dy}{dx} = \tan \theta = \frac{4h_1(L' - 2x)}{L'^2}$$

$$\tan \theta = \frac{4 \times 5 \times (45.08 - 2 \times 15)}{45.08^2}$$

$$\theta = 8.44^\circ$$

At  $q = 15 \text{ m}$ ,

$$N = V \sin \theta + H \cos \theta$$

$$= 78.62 \sin 8.44^\circ - 858.92 \sin 8.44^\circ$$

$$N = 861.25 \text{ kN}$$

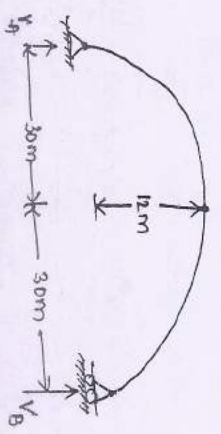
$$Q = V \cos \theta - H \sin \theta$$

$$= 78.62 \cos 8.44^\circ - 858.92 \sin 8.44^\circ$$

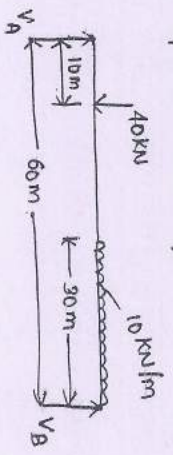
$$Q = -48.30 \text{ kN}$$

\* A 3-hinged symmetric parabolic arch of span 60 m and rise 12 m is

subjected to a concentrated load of 40 kN acting at 10 m from the left support & uniformly distributed load of intensity 10 kN/m acting over its entire right half portion. Draw the B.M.D.



The arch with its loading is shown in figure and the equivalent beam is



$$V_A + V_B = 40 + 10 \times 30 = 340 \text{ kN} \quad (1)$$

$$M_B = 0, \quad V_A \times 60 - 40(50) - 10 \times 30 \times 15 = 0$$

$$V_A = \frac{40(50) + 10 \times 30 \times 15}{60} = 108.33 \text{ kN}$$

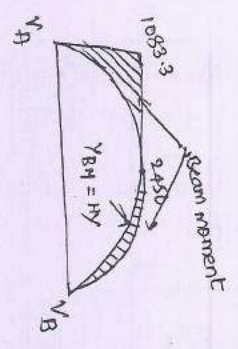
$$\therefore V_A = 108.33 \text{ kN}$$



$$\therefore V_B = 231.67 \text{ KN}$$

$$M_D = 108.33 \times 10 = 1083.3$$

$$M_C = 108.33 \times 30 - 40 \times 20 = 2450 \text{ KNm}$$



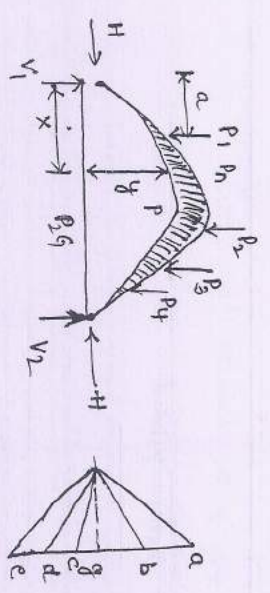
$\therefore$  At the center of the span,  $R_M$  in the arch should be zero  
 $H \times 12 = 2450$   
 $H = 204.167 \text{ KN}$

Hence, a parabola with  $V_{RB} = H_y$  is drawn over this diagram. The dip curve diagram is the B.M. diagram for the arch. At the crown hinge, the two diagrams cross each other, the B.M. at the hinge is zero.

### Eddy's Problem:-

It states that Bending moment at any point (section) of an arch is proportional to the vertical distance between the linear arch and the center line of the actual arch.

Proof:-



Consider a section 'x' which is at a distance from the point A. The distance of arch at the section 'y' represents the polygon portrays the B.M.D. to some scale the B.M. due to external on the section 'x' is given by the vertical intercepts 'p', 'q', 'r'.

Let 1 cm = p be the scale to which arch is drawn (load diagram - scale)

1 cm = q be the scale of load diagram.

The bending moment at section 'p' is given by

$$M_p = V_1 x + W_1 (x-a) - H y$$

$$= Mx - Hy$$

where  $Mx = V_1 x - W_1 (x-a)$  = usual B.M. at the section due to load & weight from the figure, we have,

$$Mx = (P_1 P_2) \times \text{scale of B.M.D.}$$

$$= P_1 P_2 (P \cdot q \cdot r^2)$$

The scale will be 'p.q.r'. This is when the distance of from the load line is 'r'.

$$H \times y = (P_1 P_2) \times \text{scale of B.M.D.}$$

2.  $P_1 P_2$  (parabolic arch)

$$M_p = \Delta U - HY$$

$$= P_1 P_2 (P_1 \alpha \cdot \gamma) - P_2 (P_1 \alpha \cdot \gamma)$$

$$= (P_1 \alpha \cdot \gamma) (P_1 P_2 - P_2)$$

$$= (P_1 P_1) (P_1 \alpha \cdot \gamma)$$

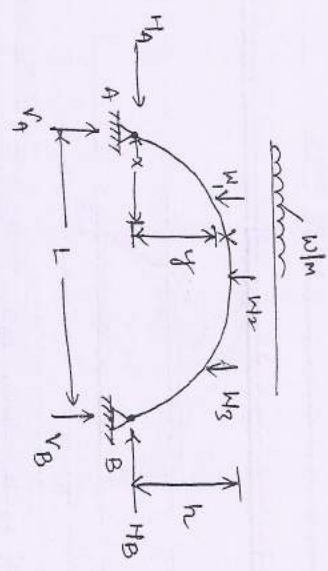
where  $P_1$  is the ordinate b/w the linear arch and the actual arch.

Hence proved.

## TWO-HINGED ARCHES

\* Two hinged arches have degree of indeterminacy one.

\* The analysis is subjected to vertical & parabolic arches only.



\* A typical two hinged arch is shown in figure. There are

two reaction components  $H_A$ ,  $V_A$ ,  $H_B$  and  $V_B$  where  $\alpha$  number of equations of equilibrium are only three. Normally arches are subjected to vertical forces only. Hence, equilibrium of forces in the horizontal directions gives,  $H_A = H_B$ , say  $H$

The horizontal force  $H$  may be taken as the redundant force. The expression for horizontal thrust may be found by any one of the following two methods.

i) First theorem of Castigliano

ii) Unit load method

\* First theorem of Castigliano [Strain energy Principle].

$$M_x = M' - HY$$

$\therefore$  Therefore, strain energy stored in the arch,  $U = \int \frac{M_x^2 ds}{2EI}$

$$= \int (M' - HY)^2 \left( \frac{ds}{2EI} \right)$$



According to Castigliano's first theorem,

$$\frac{dU}{dH} = \text{Horizontal displacement of one end w.r.t to the other}$$

$$= 0 \quad (\text{since supports are unyielding})$$

$$\therefore \frac{d}{dH} \left( M_1 - HY \right) \frac{dS}{2EI} = 0$$

$$\int 2(M_1 - HY)(-Y) \frac{dS}{2EI} = 0$$

$$\int (-M_1 Y + HY^2) \frac{dS}{EI} = 0$$

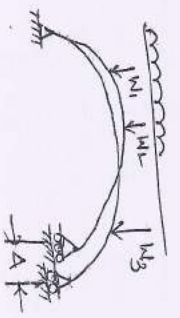
$$\int M_1 Y \frac{dS}{EI} = \int HY^2 \frac{dS}{EI}$$

$$\int M_1 Y \frac{dS}{EI} = H \int Y^2 \frac{dS}{EI}$$

$$\therefore H = \frac{\int M_1 Y \times \frac{dS}{EI}}{\int Y^2 \frac{dS}{EI}}$$

\* Unit Load Method [Consistent Deformation Method] :-

Exploiting hinged end B by storage, we get the basic determinate structure as shown in figure. In this structure,  $H=0$ . Hence,



The BM is the same as the beam moment  $M_1$  let the BM due to unit horizontal load at B be  $m'$ . Hence, at any section  $P(x, y)$  the BM due to unit load 'm' is  $m = y$

$$= \int M_1 m \left( \frac{dS}{EI} \right) = \int M_1 y \left( \frac{dS}{EI} \right)$$

Now, consider the inward displacement of the roller end when the  $H$  is applied as shown in figure. The bending moment due to  $P(x, y)$  is  $HY$ , since the bending moment due to unit load is  $y$ .

$\therefore$  Horizontal displacement of roller

$$= \int HYxy \left( \frac{dS}{EI} \right) = H \int Y^2 \left( \frac{dS}{EI} \right)$$

$\therefore$  For consistency condition, the support of arch must be vertical. Equations must be equal. Thus

$$\int M_1 y \left( \frac{dS}{EI} \right) = H \int Y^2 \left( \frac{dS}{EI} \right)$$

$$H = \frac{\int M_1 y \left( \frac{dS}{EI} \right)}{\int Y^2 \left( \frac{dS}{EI} \right)}$$

Analysis of 2-hinged circular arches:-

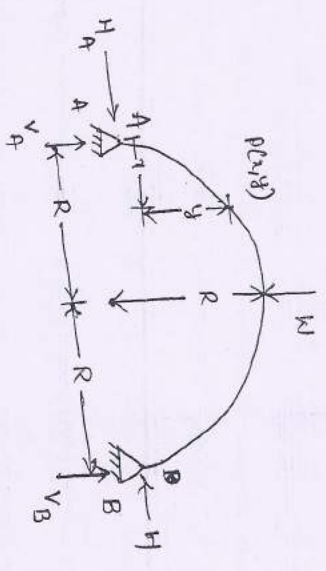
$$y = h - (R - R \cos \theta) = h - R(1 - \cos \theta)$$

$$ds = R d\theta$$

$$N = V \sin \theta + H \cos \theta$$

$$Q = V \cos \theta - H \sin \theta$$

The horizontal thrust developed in a semi circular arch subjected to a concentrated load W at the crown



Consider an arch shown in figure.

Due to symmetry,

$$V_A = V_B = \frac{W}{2}$$

$$y = R \cos \theta$$

$$x = \frac{L}{2} - R \sin \theta = \frac{2R}{2} - R \sin \theta = R(1 - \sin \theta)$$

$$\therefore \frac{L}{2} = R$$

$$M^1 = V_A x = \frac{W}{2} x = \frac{W}{2} R(1 - \sin \theta)$$

$$H = \frac{\int M^1 y \left( \frac{ds}{EI} \right)}{\int y^2 \left( \frac{ds}{EI} \right)} = \frac{\int M^1 y ds}{\int y^2 ds} \quad (\because EI \text{ is constant})$$

$$(\because ds = R d\theta)$$

$$\int M^1 y ds = 2 \int_0^{\pi/2} \left( \frac{W}{2} \right) R(1 - \sin \theta) R \cos \theta R d\theta$$

$$= WR^3 \int_0^{\pi/2} (1 - \sin \theta) \cos \theta d\theta$$

$$= WR^3 \int_0^{\pi/2} (1 - \sin \theta) \cos \theta d\theta$$

Let,  $(1 - \sin \theta) = t$

then,  $dt = -\cos \theta d\theta$

$$\therefore \int (1 - \sin \theta) \cos \theta d\theta = \int t(-dt) = -\frac{t^2}{2}$$

$$= -\frac{1}{2} [(1 - \sin \theta)^2]$$

$$\therefore M^1 y ds = WR^3 \left[ -\frac{1}{2} (1 - \sin \theta)^2 \right]_0^{\pi/2}$$

$$= \frac{WR^3}{2} [-(1-0) - (1-0)]$$

$$= \frac{WR^3}{2}$$

now, consider the denominator of the expression for H.

$$\int y^2 ds = 2 \int_0^{\pi/2} R^2 \cos^2 \theta R d\theta$$

$$= 2R^3 \int_0^{\pi/2} (1 + \cos 2\theta) d\theta$$

$$= R^3 \left[ \theta + \frac{1 + \sin 2\theta}{2} \right]_0^{\pi/2}$$

$$= R^3 \left[ \frac{\pi}{2} + \frac{1}{2} + 0 - 0 - \frac{1}{2} - 0 \right]$$

$$= \frac{\pi R^3}{2}$$

NOTE:-

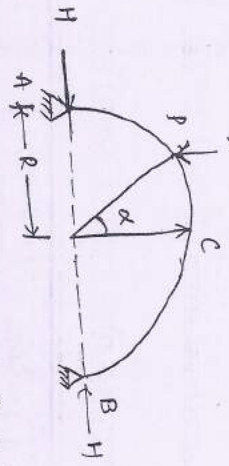
The horizontal thrust for a semi-circular arch is indigo of its radius

$$\therefore H = \frac{\int M^1 y ds}{\int y^2 ds} = \frac{WR^3}{2} \times \frac{2}{\pi R^3} = \frac{W}{\pi}$$

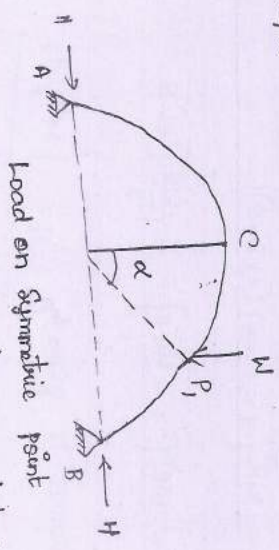
$$\therefore H = \frac{W}{\pi}$$



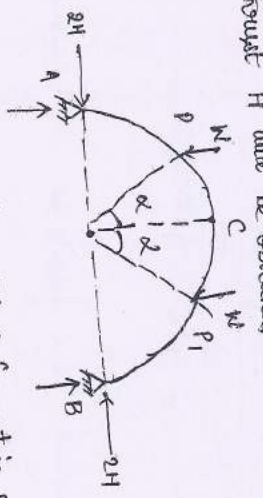
when a load  $W$  acts at a point  $P$  as shown in figure assume uniform flexural rigidity.



Let the horizontal thrust produced be  $H$ . If  $P_1$  is the symmetric point of  $P$  and if  $W$  acts at this point, naturally the horizontal thrust will be  $H$ .



hence, in the case shown in below figure, the horizontal thrust will be  $2H$ . This arch will be analyzed to get  $2H$  and then horizontal thrust  $H$  will be obtained.



Load at the point & symmetric point

$$V_A = V_B = W$$

$$H = \frac{W}{2} - R \sin \theta = R - R \sin \theta$$

$$H = R(1 - \sin \theta)$$

$$Y = h - R(1 - \cos \theta) = R - R \cos \theta$$

In the portion  $\theta = 0$  to  $\alpha$

$$M' = Wx - WR(\sin \theta - \sin \theta)$$

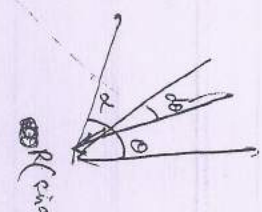
$$= WR(1 - \sin \theta) - WR(\sin \theta - \sin \theta)$$

$$= WR(1 - \sin \alpha)$$

For the portion,  $\theta = \alpha$  to  $\pi/2$ ,

$$M' = Wx = WR(1 - \sin \theta)$$

$$\therefore \text{Horizontal thrust} = \frac{\int M' y \left( \frac{dy}{dx} \right)}{\int y^2 \left( \frac{dy}{dx} \right)} = \frac{\int M' y ds}{\int y^2 ds}$$



$$\int M' y ds = 2 \int_0^\alpha WR(1 - \sin \alpha) R \cos \theta d\theta + 2 \int_\alpha^{\pi/2} WR(1 - \sin \theta) R \cos \theta d\theta$$

$$= 2WR^2(1 - \sin \alpha) \int_0^\alpha \cos \theta d\theta + 2WR^2 \int_\alpha^{\pi/2} (1 - \sin \theta) \cos \theta d\theta$$

$$= 2WR^2(1 - \sin \alpha) [\sin \theta]_0^\alpha + 2WR^2 - \left[ \frac{(1 - \sin \theta)^2}{2} \right]_\alpha^{\pi/2}$$

$$= 2WR^2(1 - \sin \alpha) \sin \alpha - WR^2 [1 - 0]^2 - (1 - \sin \alpha)^2 R^2$$

$$= 2WR^2(1 - \sin \alpha) \sin \alpha + WR^2 [1 - \sin^2 \alpha]$$

$$= WR^2(1 - \sin \alpha) [2 \sin \alpha + 1 + \sin^2 \alpha]$$

$$= WR^2(1 - \sin \alpha) (1 + \sin \alpha)$$

$$= WR^2 \cos^2 \alpha$$

$$\int y^2 ds = 2 \int_0^\alpha R^2 \cos^2 \theta R d\theta + 2 \int_\alpha^{\pi/2} R^2 \cos^2 \theta R d\theta = 2R^3 \int_0^\alpha \cos^2 \theta d\theta + 2R^3 \int_\alpha^{\pi/2} \cos^2 \theta d\theta$$

$$= R^3 \left[ \frac{\theta + \sin 2\theta}{2} + 0 - 0 \right] = \frac{\pi R^3}{2}$$

$$\therefore \int y^2 ds = \frac{\pi R^3}{2}$$

Horizontal thrust =  $\int y^x ds$

$$= \int y^x ds$$

$$= \int MR^3 \cos^3 \alpha \times \left(\frac{2R}{\pi R^3}\right)$$

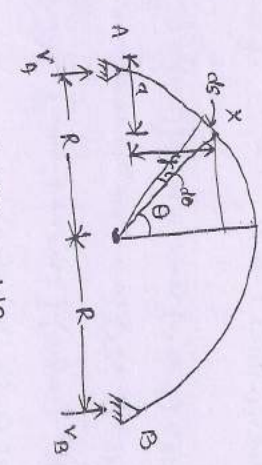
$$= \frac{2M}{\pi} \cos^3 \alpha$$

i.e.  $2H = \frac{2M}{\pi} \cos^3 \alpha$

$$\therefore H = \frac{M}{\pi} \cos^3 \alpha$$

when load is at crown,  $\alpha = 0$ ,  $H = \frac{M}{\pi}$

semi-circular arch of radius R is subjected to a U.D.L of unit length over the entire span. Assuming EI to be constant, determine horizontal thrust



$$V_A = V_B = \frac{1}{2} \times W \times 2R = WR$$

$$H = \frac{2}{\pi} - R \sin \theta = \frac{2R}{\pi} - R \sin \theta = R(1 - \sin \theta)$$

$$y = h - R(1 - \cos \theta) = R - R(1 - \cos \theta) = R \cos \theta$$

$$x = R(1 - \sin \theta) \quad \therefore y = R \cos \theta$$

$$ds = R d\theta$$

$$= WR \times \frac{2R}{\pi}$$

$$= WR \times R(1 - \sin \theta) - \left[ \frac{W}{2} (R(1 - \sin \theta))^2 \right]$$

$$= WR^2 (1 - \sin \theta) - \frac{WR^2}{2} (1 - \sin \theta)^2$$

$$= WR^2 \left[ 1 - \sin \theta - \frac{(1 - \sin \theta)^2}{2} \right]$$

$$= \frac{WR^2}{2} (1 - \sin \theta) [2 - (1 - \sin \theta)]$$

$$= \frac{WR^2}{2} (1 - \sin \theta) (1 + \sin \theta)$$

$$= \frac{WR^2}{2} (1 - \sin^2 \theta) = \left( \frac{WR^2}{2} \right) \cos^2 \theta$$

Horizontal thrust is given by  $H = \frac{\int M^1 y ds}{\int y^x ds}$

$$\therefore \text{Numerator } \int M^1 y ds = 2 \int_0^{\pi/2} \left( \frac{WR^2}{2} \right) \cos^2 \theta \times R \cos \theta \times R d\theta$$

$$= WR^4 \int_0^{\pi/2} \cos^3 \theta d\theta$$

New, consider  $I = \int \cos^3 \theta d\theta = \int (1 - \sin^2 \theta) \cos \theta d\theta$

Let  $t = \sin \theta$   
 $dt = \cos \theta d\theta$

$$\therefore I = \int (1 - t^2) dt$$

$$= \left[ t - \frac{t^3}{3} \right] = \sin \theta - \frac{\sin^3 \theta}{3}$$



$$\therefore \text{Numerator} = MR \int_0^{\pi/2} \left[ \sin \theta - \frac{\sin 2\theta}{2} \right] d\theta$$

$$= MR^4 \left[ 1 - \frac{1}{3} \right]$$

$$= \frac{2}{3} MR^4$$

$$\text{Denominator} = 2 \int_0^{\pi/2} r^2 \cos^2 \theta \, r \, d\theta$$

$$= 2R^3 \int_0^{\pi/2} (1 + \cos 2\theta) d\theta$$

$$= R^3 \left[ \theta + \frac{\sin 2\theta}{2} \right]_0^{\pi/2}$$

$$= R^3 \left[ \frac{\pi}{2} + 0 - 0 - 0 \right]$$

$$= \frac{\pi R^3}{2}$$

$$\therefore H = \frac{2}{3} MR^4 \times \frac{2}{\pi R^3}$$

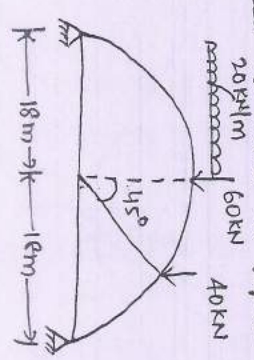
$$\therefore H = \frac{4}{3} \left( \frac{MR}{\pi} \right)$$

2. Determine the horizontal thrust developed in a semi-circular circular arch subjected to a UDL on only one-half of the arch. EI is constant

(4)  $\therefore 2H = \frac{4}{3} \left( \frac{WR}{\pi} \right)$

$$\therefore H = \frac{2}{3} \left( \frac{WR}{\pi} \right)$$

\* Determine the horizontal thrust developed in the semi-circular arch of radius 18 m loaded as shown in figure.



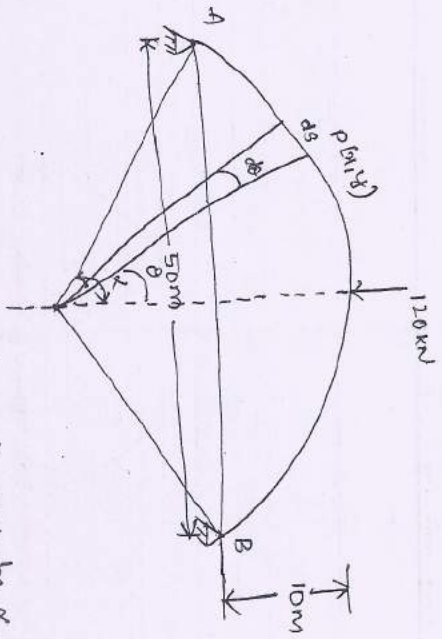
Using the results obtained in standard cases, the thrust developed due to each case is determined separately and superposed get the horizontal thrust in this case

$$H = \left[ \frac{2}{3} \frac{WR}{\pi} \right] + \frac{8W}{\pi} + \frac{W}{\pi} \cos^2 \alpha$$

$$= \left[ \frac{2}{3} \times 20 \times 18 \right] + \frac{60}{\pi} + \left[ \frac{40}{\pi} \cos^2 45^\circ \right]$$

$$= 105.41 \text{ kN}$$

the general expression for the deflection of a beam subjected to a concentrated load  $M$  at the end. Evaluate its value for the arch shown in figure. Constant flexural rigidity.



Let the kerf. Central angle of the arch be  $\alpha$ . Considering elemental length  $ds$  at an angular distance  $\theta$  from the vertical axis through the crown and taking A as origin, we get

$$x = R(\sin\alpha - \sin\theta)$$

$$y = R(\cos\theta - \cos\alpha)$$

$$ds = R d\theta$$

Reaction,  $V_A = V_B = \frac{W}{2}$

$$M'_1 = \frac{W}{2} \times x = \frac{W}{2} \times R(\sin\alpha - \sin\theta)$$

Since, EI is constant,

$$H = \frac{\int M'_1 y \frac{ds}{EI}}{\int y^2 \frac{ds}{EI}} = \frac{\int M'_1 y ds}{\int y^2 ds}$$

$$\text{Numerator (N)} = \int M'_1 y ds$$

$$= \int \left(\frac{W}{2}\right) R(\sin\alpha - \sin\theta) R(\cos\theta - \cos\alpha) R d\theta$$

$$= \frac{WR^3}{2} \left[ 2 \int_0^\alpha (\sin\alpha - \sin\theta)(\cos\theta - \cos\alpha) d\theta \right]$$

$$= WR^3 \int_0^\alpha (\sin\alpha \cos\theta - \sin\alpha \cos\alpha - \sin\theta \cos\theta + \sin\theta \cos\alpha) d\theta$$

$$= WR^3 \int_0^\alpha (\sin\alpha \cos\theta - \sin\alpha \cos\alpha - \frac{\sin 2\theta}{2} + \sin\theta \cos\alpha) d\theta$$

$$= WR^3 \left[ \sin\alpha \sin\theta - \theta \sin\alpha \cos\alpha - \frac{1}{2} \left( -\frac{\cos 2\theta}{2} \right) + (-\cos\theta) \cos\alpha \right]_0^\alpha$$

$$= WR^3 \left[ \sin^2 \alpha - \alpha \sin\alpha \cos\alpha + \frac{1}{4} (\cos 2\alpha - 1) - \cos\alpha (\cos\alpha - 1) \right]$$

$$= WR^3 \left[ \sin^2 \alpha - \alpha \sin\alpha \cos\alpha + \frac{1}{4} (2\cos^2 \alpha - 1 - 1) - \cos\alpha (\cos\alpha - 1) \right]$$

$$= WR^3 \left[ \sin^2 \alpha - \alpha \sin\alpha \cos\alpha - \frac{1}{2} \sin^2 \alpha - \cos^2 \alpha + \cos\alpha \right]$$

$$= WR^3 \left[ \frac{\sin^2 \alpha}{2} - \cos\alpha (\cos\alpha - 1 + \alpha \sin\alpha) \right]$$

$$N = WR^3 \left[ \sin^2 \alpha - 2 \cos\alpha (\cos\alpha - 1 + \alpha \sin\alpha) \right]$$

$$\text{Denominator (D)} = \int y^2 ds$$

$$= \int R^2 (\cos\theta - \cos\alpha)^2 R d\theta$$

$$= R^3 \int_0^\alpha (\cos^2 \theta - 2 \cos\alpha \cos\theta + \cos^2 \alpha) d\theta$$



$$= 2R^3 \int_0^\alpha \left( \frac{1+\cos\theta}{2} - 2\cos\theta \cos\theta + \cos^2\theta \right) d\theta$$

$$= 2R^3 \left[ \frac{\theta}{2} + \frac{\sin 2\theta}{2} - 2\cos\theta \sin\theta + \theta \cos^2\theta \right]_0^\alpha$$

$$= 2R^3 \left[ \frac{\alpha}{2} + \frac{\sin 2\alpha}{4} - 2\cos\alpha \sin\alpha + \alpha \cos^2\alpha \right]$$

$$= R^3 \left[ \alpha + \frac{\sin 2\alpha}{2} - 4\cos\alpha \sin\alpha + 2\alpha \cos^2\alpha \right]$$

$$= R^3 \left[ \alpha + \frac{\sin 2\alpha}{2} - 2\sin 2\alpha + 2\alpha \cos^2\alpha \right]$$

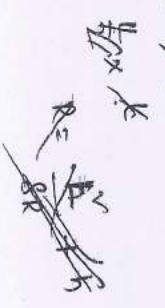
$$= R^3 \left[ \alpha - \frac{3\sin 2\alpha}{2} + 2\alpha \cos^2\alpha \right]$$

$$D = \frac{R^3}{2} [4\alpha \cos^2\alpha + 2\alpha - 3\sin 2\alpha]$$

$$\therefore H = \frac{\int M' y ds}{\int y^2 ds} = \frac{WR^3 [\sin^2\alpha - 2\cos\alpha (\cos\alpha - 1 + \alpha \sin\alpha)]}{\frac{R^3}{2} [4\alpha \cos^2\alpha + 2\alpha - 3\sin 2\alpha]}$$

$$\therefore H = \frac{W}{2} \left[ \frac{\sin^2\alpha - 2\cos\alpha (\cos\alpha - 1 + \alpha \sin\alpha)}{4\alpha \cos^2\alpha + 2\alpha - 3\sin 2\alpha} \right]$$

$\therefore$  In the given problem, from geometry of circle



$$R = \frac{1}{85} + \frac{5}{2}$$

$$= \frac{50}{8 \times 10} + \frac{10}{2} = \frac{50 \times 50}{8 \times 10} + 5 = \frac{250}{8} + 5 = 31.25 + 5$$

$$\therefore R = 36.25 \text{ m}$$

and  $R \sin \alpha = 25 \text{ m}$

$$\therefore \sin \alpha = \frac{25}{R}$$

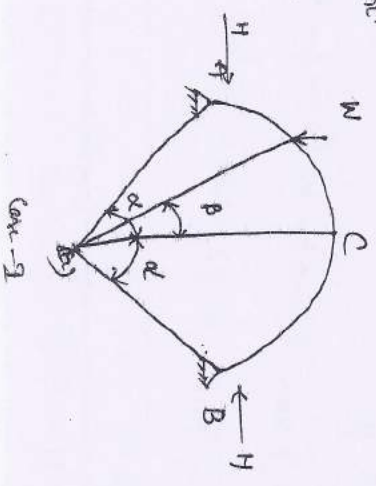
$$\alpha = \sin^{-1} \left( \frac{25}{36.25} \right) = 43.6^\circ$$

$$H = \frac{W}{2} \left[ \frac{\sin^2\alpha - 2\cos\alpha (\cos\alpha - 1 + \alpha \sin\alpha)}{4\alpha \cos^2\alpha + 2\alpha - 3\sin 2\alpha} \right]$$

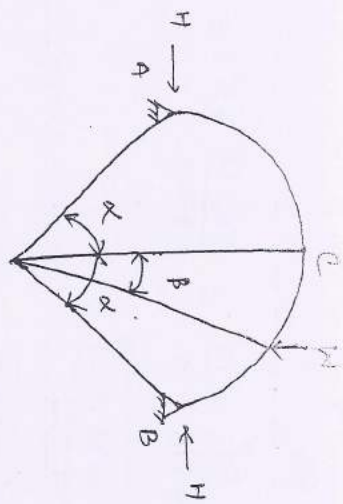
$$= \frac{120}{2} \left[ \frac{0.4756 - 1.4483(0.7241 - 1 + 0.52848)}{1.5902 + 1.5220 - 2.9964} \right]$$

$$H = 113.45 \text{ kN}$$

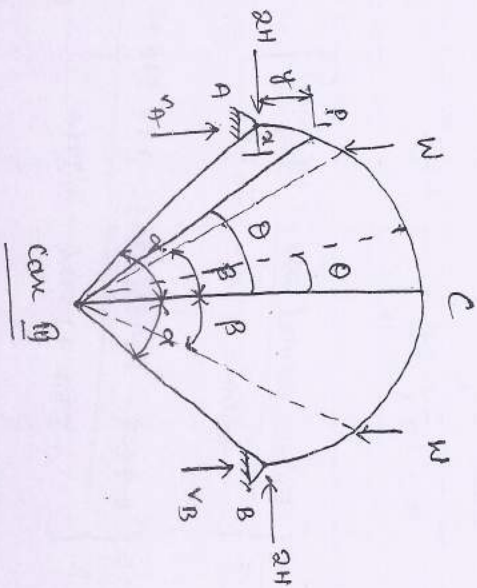
\* Determine the expression for horizontal thrust in a segmental arch to a temperature load at angular distance  $\theta$  as shown in figure. EI to be constant.



Ex-9



Case - II



Case III

$$V_A = V_B = W$$

Case III

$$\theta = 0 \text{ to } \beta$$

$$M' = W (R \sin \alpha - R \sin \theta) - W (R \sin \beta - R \sin \theta)$$

=  $WR (\sin \alpha - \sin \beta)$  which is constant w.r.t  $\theta$

Case III

$$\theta = \beta \text{ to } \alpha$$

$$M' = WR (\sin \alpha - \sin \theta)$$

$$\Delta H = \frac{\int M' y \frac{ds}{EI}}{\int y^2 \frac{ds}{EI}} = \frac{\int M' y ds}{\int y^2 ds} \quad (\because EI \text{ is constant})$$

$$\text{Numerator } (N) = 2 \int_0^\beta WR (\sin \alpha - \sin \beta) R (\cos \theta - \cos \alpha) R d\theta$$

$$+ 2 \int_\beta^\alpha WR (\sin \alpha - \sin \theta) R (\cos \theta - \cos \alpha) R d\theta$$

$$= N_1 + N_2$$

$$\therefore N_1 = 2WR^3 \int_0^\beta (\sin \alpha - \sin \beta) (\cos \theta - \cos \alpha) d\theta$$

$$= 2WR^3 (\sin \alpha - \sin \beta) \int_0^\beta (\cos \theta - \cos \alpha) d\theta$$

$$= 2WR^3 (\sin \alpha - \sin \beta) \left[ \sin \theta - \theta \cos \alpha \right]_0^\beta$$

$$= 2WR^3 (\sin \alpha - \sin \beta) \left[ \sin \beta - \beta \cos \alpha \right]$$

$$\therefore N_2 = 2WR^3 \int_\beta^\alpha (\sin \alpha - \sin \theta) (\cos \theta - \cos \alpha) d\theta$$

$$= 2WR^3 \left[ \sin \alpha \sin \theta - \theta \sin \alpha \cos \alpha + \frac{1}{4} (\cos \theta - \cos \alpha) \cos \theta \right]_\beta^\alpha$$

as in previous example

$$= 2WR^3 \left[ \sin \alpha \sin \theta - \theta \sin \alpha \cos \alpha + \frac{1}{4} (2 \cos^2 \theta - 1) - \cos \alpha \cos \theta \right]_\beta^\alpha$$



$$= 2MR^3 \left[ \sin^2 \alpha - \sin \alpha \sin \beta - \alpha \sin \alpha \cos \alpha + \beta \sin \alpha \cos \beta + \frac{1}{4} (2 \cos^2 \alpha - 1 - 2 \cos^2 \beta + 1) - \cos^2 \alpha + \cos \alpha \sin \beta \right]$$

$$N_2 = 2MR^3 \left[ \sin^2 \alpha - \sin \alpha \sin \beta - \alpha \sin \alpha \cos \alpha + \beta \sin \alpha \cos \beta + \frac{\cos^2 \alpha}{2} - \frac{\cos^2 \beta}{2} - \cos^2 \alpha + \cos \alpha \cos \beta \right]$$

$$\therefore N_1 + N_2 = 2MR^3 (\sin \alpha - \sin \beta) (\sin \beta - \beta \cos \alpha) + 2MR^3 \left[ \sin^2 \alpha - \sin \alpha \sin \beta - \alpha \sin \alpha \cos \alpha + \beta \sin \alpha \cos \beta + \frac{\cos^2 \alpha}{2} - \frac{\cos^2 \beta}{2} - \cos^2 \alpha + \cos \alpha \cos \beta \right]$$

$$= 2MR^3 \left[ \sin \alpha \sin \beta - \beta \sin \alpha \cos \alpha - \sin^2 \beta - \beta \sin \beta \cos \alpha + \cos \alpha \cos \beta + \frac{\sin^2 \alpha}{2} - \frac{\sin^2 \beta}{2} \right]$$

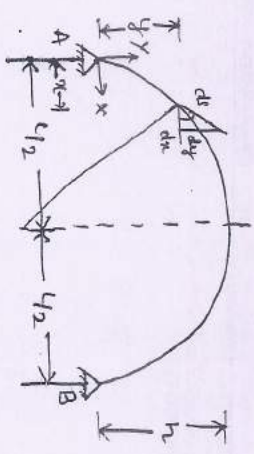
$$= MR^3 \left[ \sin^2 \alpha - \sin^2 \beta - 2 \cos \alpha (\cos \alpha + \alpha \sin \alpha - \beta \sin \beta - \cos \beta) \right]$$

$$N = MR^3 \left[ \sin^2 \alpha - \sin^2 \beta - 2 \cos \alpha (\cos \alpha - \cos \beta + \alpha \sin \alpha - \beta \sin \beta) \right]$$

$$\therefore 2H = \frac{N}{D} = \frac{2MR^3 \left[ \sin^2 \alpha - \sin^2 \beta - 2 \cos \alpha (\cos \alpha - \cos \beta + \alpha \sin \alpha - \beta \sin \beta) \right]}{R^3 \left[ 4 \alpha \cos^2 \alpha + 2 \alpha - 3 \sin 2 \alpha \right]}$$

$$\therefore H = \frac{M}{D} \left[ \sin^2 \alpha - \sin^2 \beta - 2 \cos \alpha (\cos \alpha - \cos \beta + \alpha \sin \alpha - \beta \sin \beta) \right] \frac{R^3}{4 \alpha \cos^2 \alpha + 2 \alpha - 3 \sin 2 \alpha}$$

\* Analysis of Parabolic Arch:-



Consider a typical parabolic arch shown in figure. The left springing point as the origin, the equation of the parabola is

$$y = \frac{4h}{L^2} x(L-x)$$

where \$L\$ is the span and \$h\$ is the rise. Let \$P(x,y)\$ be a point on the arch and \$ds\$ be the elemental length. Let \$\theta\$ be the slope of the arch horizontal at \$P\$. Then,

$$\frac{dy}{dx} = \tan \theta$$

\$\delta ds = dx \sec \theta\$  
Horizontal thrust \$H\$ is given by

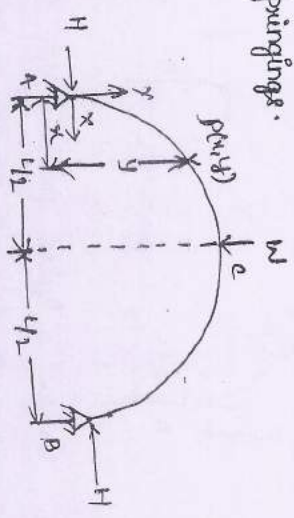
If \$q\$ is varying such that at any point \$H = \frac{\int M' y \left( \frac{ds}{dx} \right)}{\int y' \left( \frac{ds}{dx} \right)} = \frac{\int M' y \left( \frac{dx}{ds} \right)}{\int y' \left( \frac{dx}{ds} \right)}

$$H = \frac{\int M^1 y \frac{dM^1}{EI_0}}{\int y^2 \frac{dM^1}{EI_0}} = \frac{\int M^1 y dx}{\int y^2 dx} \quad (\text{since, } EI_0 \text{ is a constant})$$

is, in a parabolic arch with variation of moment of inertia in the

$$H = \frac{\int M^1 y dx}{\int y^2 dx}$$

hinged parabolic arch of span  $L$  and rise  $h$  carries a concentrated load  $W$  at the crown. Determine the expression for horizontal thrust at springings.



Consider the typical arch shown in figure. Due to symmetry, reactions at A and B are the same and its value is equal to  $\frac{W}{2}$ .

At any point P(x, y) beam moment ( $M^1$ ) is given by

$$M^1 = \left(\frac{W}{2}\right)x$$

and in the parabola,  $y = \frac{4hx(L-x)}{L^2}$

Now,  $H = \frac{\int M^1 y dx}{\int y^2 dx}$

Considering the numerator (N),  $\int M^1 y dx = \int \left(\frac{W}{2}\right)x y dx$

$$= \left(\frac{2Wh}{L^2}\right) 2 \int_0^{L/2} (Lx^2 - x^3) dx$$

$$= \frac{4Wh}{L^2} \left[ \frac{Lx^3}{3} - \frac{x^4}{4} \right]_0^{L/2}$$

$$= \frac{4Wh}{L^2} \left[ \frac{L^4}{24} - \frac{L^4}{64} \right]$$

$$= 4WhL^2 \left[ \frac{8-9}{192} \right]$$

$$N = \frac{5}{48} WhL^2 //$$

Denominator (D) is given by

$$D = \int y^2 dx$$

$$= \int \left[ \frac{4hx(L-x)}{L^2} \right]^2 dx$$

$$= \left(\frac{16h^2}{L^4}\right) 2 \left[ \int_0^{L/2} x^2(L^2 + x^2 - 2Lx) dx \right]$$

$$= \frac{32h^2}{L^4} \left[ \int_0^{L/2} (L^2x^2 + x^4 - 2Lx^3) dx \right]$$

$$= \frac{32h^2}{L^4} \left[ \frac{L^2x^3}{3} + \frac{x^5}{5} - \frac{2Lx^4}{4} \right]_0^{L/2}$$

$$= \frac{32h^2}{L^4} L^5 \left[ \frac{1}{24} - \frac{1}{32} + \frac{1}{160} \right]$$

$$= 32h^2 L \left[ \frac{30-15+3}{480} \right]$$



$$D = \left(\frac{8}{15}\right) h^2 L$$

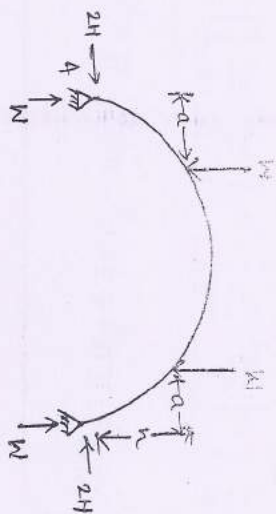
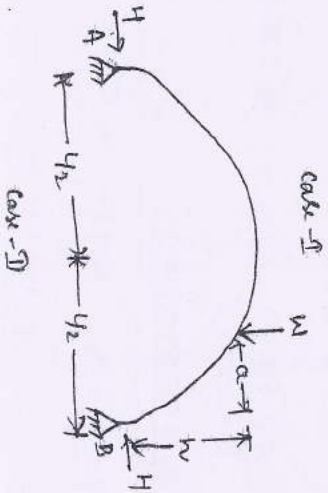
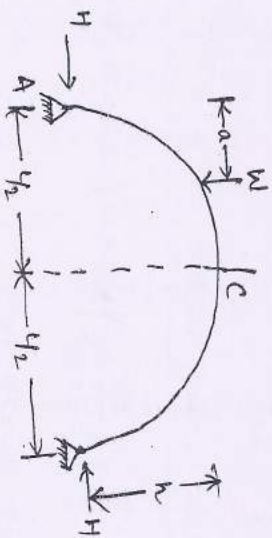
$$\therefore H = \frac{N}{D} = \frac{5}{48} \frac{M h L^2 \times \frac{15}{8 h^2 L}}{1}$$

$$\therefore H = \frac{25}{128} \left(\frac{M L}{h}\right)$$

\* Show that the horizontal thrust developed in a parabolic arch of span  $L$  and rise  $h$  subjected to a concentrated load  $W$  at a distance  $a$  from a springing is given by

$$H = \left(\frac{5}{8}\right) \left(\frac{W}{h L^3}\right) a (L-a) (L^2 + La - a^2)$$

Sol:-



In case-III, Vertical reaction at end A is  $W$

$\therefore$  Beam moment at a point  $P(x, y)$  is

$$M' = Wx, \text{ for the portion } x=0 \text{ to } a$$

$$M' = Wx - W(x-a)$$

$$= Wa \text{ for the portion of } x=a \text{ to } L/2$$

Horizontal thrust,

$$2H = \frac{\int M' y' dx}{\int y' dx}$$

Now,  $\int y' dx = \frac{8}{15} h^2 L$ , as in previous problem

$$\int M' y' dx = 2 \left[ \int_0^a Wxy' dx + \int_a^{L/2} Waxy' dx \right]$$

$$= 2 \int_0^a Wx \times \frac{4hx(L-x)}{L^2} dx + 2 \int_a^{L/2} Wa \times \frac{4hx(L-x)}{L^2} dx$$

$$= \left(\frac{8Wh}{L^2}\right) \int_0^a (Lx^2 - x^3) dx + \left(\frac{8Wh}{L^2}\right) a \int_a^{L/2} (Lx - x^2) dx$$

$$= \frac{8Wh}{L^2} \left[ \left(\frac{Lx^3}{3} - \frac{x^4}{4}\right) \Big|_0^a + a \left(\frac{Lx^2}{2} - \frac{x^3}{3}\right) \Big|_a^{L/2} \right]$$

$$\frac{1}{24} \left[ \frac{1}{3} \left( \frac{1}{2} + a \left[ \frac{L^3}{8} - \frac{L^2}{24} \right] - a^2 \left[ \frac{L^2}{2} - \frac{aL}{3} \right] \right) \right]$$

$$\left( \frac{8Mh}{L^2} \right) \frac{1}{24} \left[ 81a^3 - 6a^2L + 3aL^2 - aL^3 - 12a^2L + 8a^3 \right]$$

$$\frac{Mha}{3L^2} \left[ 2L^3 - 4a^2L + 2a^3 \right]$$

$$= \frac{Mha}{3L^2} \times 2 \left[ L^3 - 2a^2L + a^3 \right]$$

$$= \left( \frac{2}{3} \right) \left( \frac{Mha}{L^2} \right) \left[ L^3 - 2a^2L + a^3 \right]$$

$$= \frac{2}{3} \left( \frac{Mha}{L^2} \right) \left[ L(L^2 - 2a^2) - a^2(L - a) \right]$$

$$= \frac{2}{3} \left( \frac{Mha}{L^2} \right) \left[ L(L - a)(L + a) - a^2(L - a) \right]$$

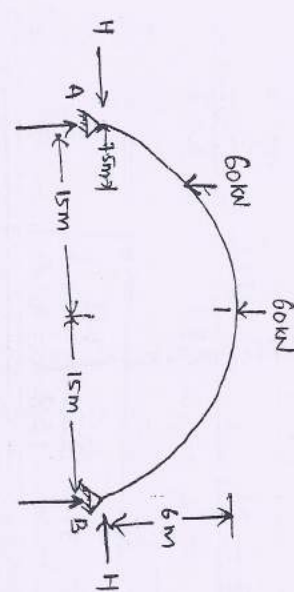
$$= \frac{2}{3} \left( \frac{Mha}{L^2} \right) (L - a) \left[ L^2 + La - a^2 \right]$$

$$\int y^2 dx = \frac{2}{3} \left( \frac{Mha}{L^2} \right) (L - a) \int (L^2 + La - a^2) \times \frac{15}{8h^2} dx$$

$$= \frac{5}{4} \left( \frac{M}{hL^2} \right) a(L - a) (L^2 + La - a^2)$$

$$\therefore H = \frac{5}{8} \left( \frac{M}{hL^2} \right) a(L - a) (L^2 + La - a^2)$$

two point loads, each 60kN, acting at 7.5m and 15m from the left end, respectively. The moment of inertia varies as the square of slope. Determine the horizontal thrust and maximum positive and negative moments in the arch ribs.



The arch is shown in figure. Let the horizontal thrust due to 60kN load at 7.5m from A be  $H_1$  and that due to 60kN load at 15m from B be  $H_2$ . Then,

$$H = H_1 + H_2$$

$$\therefore H = \frac{5}{8} \times \frac{M}{hL^2} a(L - a) (L^2 + La - a^2)$$

where  $M = 60kN$ ,  $h = 6m$ ,  $L = 30m$  and  $a = 7.5m$ .

$$H_1 = \frac{5}{8} \times \left( \frac{60}{6 \times 30^2} \right) \times 7.5 \times (30 - 7.5) \times (30^2 + 30 \times 7.5 - (7.5)^2)$$

$$H_1 = 41.748 \text{ kN}$$

$$H_2 = \frac{5}{8} \times \left( \frac{60}{6 \times 30^2} \right) \times 15 \times (30 - 15) \times (30^2 + 30 \times 15 - 15^2)$$

$$H_2 = 58.594 \text{ kN}$$

$$\therefore H = H_1 + H_2 = 41.748 + 58.594 = 100.342 \text{ kN}$$

$$\therefore H = 100.342 \text{ kN}$$



consider the member from the right end point. Now,

$$V_A + V_B = 120$$

$$\sum M_B = 0,$$

$$V_A \times 30 - 60 \times (30 - 7.5) - 60 \times 15 = 0$$

$$V_A \times 30 = 60 \times 22.5 + 60 \times 15$$

$$\therefore V_A = 75 \text{ kN}$$

$$\therefore V_B = 120 - 75 = 45 \text{ kN}$$

$$\therefore V_B = 45 \text{ kN}$$

At 7.5m from the left spanning point A,

$$M_x = 8 V_A \times x - 4x \times x$$

$$= 75 \times 7.5 - 100.342 \times 4.5$$

$$M_x = 110.961 \text{ kNm}$$

$$y = \frac{4hx(L-x)}{L^2}$$

$$= \frac{4 \times 7.5 \times 7.5 (30 - 7.5)}{30^2}$$

$$= 4.5$$

$$(\therefore y = h = 6)$$

$$M_x = V_B \times 15 - 4x \times y$$

$$= 45 \times 15 - 100.342 \times 6$$

$$= 72.948 \text{ kNm}$$

$\therefore$  Max. negative moment occurs in the right half portion, measuring  $x$  from B.

$$M_x = 45x - 100.342y$$

$$= 45x - 100.342 \left[ \frac{4x \times 6 \times x (30 - x)}{30^2} \right]$$

$$= 45x - 2.646 (30x - x^2)$$

$$\frac{dM}{dx} = 0 \Rightarrow 45 - 2.646 \times 2 (30 - 2x)$$

$$\therefore x = 6.59 \text{ m}$$

$\therefore$  maximum negative moment =  $45(6.59) - 2.646 \times 2 \times 30 \times 6$

$$= 45x - 100.342y$$

$$= 45 \times 6.59 - 100.342 \left[ \frac{4(6.59)(6.59)(30 - 6.59)}{30^2} \right]$$

$$= -116.28 \text{ kNm}$$

\* Determine the horizontal reaction in the above problem carrying integration.

$$\text{Sol} \quad V_A = 75 \text{ kN} \quad \& \quad V_B = 45 \text{ kN}$$

For  $x = 0$  to  $7.5$  m from A,  $M'_x = V_A \times x = 75x$

For  $x = 7.5$  to  $15$  m from A,  $M'_x = V_A \times x - 60(x - 7.5)$

$$= 75x - 60x + 450$$

$$= 15x + 450$$

For  $x = 0$  to  $15$  from B,  $M'_x = V_B \times x = 45x$

$$\therefore y = \frac{4hx(L-x)}{L^2} = \frac{4(6)x(30-x)}{30^2} = \frac{24x(30-x)}{900}$$

$$\therefore \int M'_y dx = \int_0^{7.5} 75x y dx + \int_{7.5}^{15} (15x + 450) y dx + \int_{7.5}^{15} 45x y dx$$

$$= \int_0^{7.5} 75x \times \frac{24}{900} \times (30x - x^2) dx + \int_{7.5}^{15} (15x + 450) \times \frac{24}{900} \times (30x - x^2) dx$$

$$+ \int_{7.5}^{15} 45x \times \frac{24}{900} \times (30x - x^2) dx$$

$$7.5x \times \frac{24}{900} \times (30x - x^2) dx + \int_{7.5}^{15} (15x + 45) \times \frac{24}{900} \times (30x - x^2) dx$$

$$+ \int_{7.5}^{15} 45x \times \frac{24}{900} \times (30x - x^2) dx$$

$$2(30x^2 - x^3) dx + \int_{7.5}^{15} \frac{24}{900} \times (15x + 45)(30x - x^2) dx$$

$$+ \int_{7.5}^{15} 1.2(30x^2 - x^3) dx$$

$$\int_{7.5}^{15} (30x^2 - x^3) dx + \frac{24}{900} \int_{7.5}^{15} (15x + 45)(30x - x^2) dx$$

$$+ 1.2 \int_{7.5}^{15} (30x^2 - x^3) dx$$

$$\left[ 30 \times \frac{x^3}{3} - \frac{x^4}{4} \right]_0^{7.5} + 1.2 \left[ 30 \times \frac{x^3}{3} - \frac{x^4}{4} \right]_0^{15}$$

$$+ \frac{24}{900} \int_{7.5}^{15} (450x^2 - 15x^3 + 13500x - 450x^2) dx$$

$$2 \left[ 30 \times \frac{7.5^3}{3} - \frac{7.5^4}{4} \right] + 1.2 \left[ 30 \times \frac{15^3}{3} - \frac{15^4}{4} \right]$$

$$+ \frac{24}{900} \left[ -15 \left( \frac{15^4}{4} \right) + 13500 \left( \frac{15}{2} \right) \right]$$

$$= 32167.964 + \frac{24}{900} \left[ -15 \left( \frac{15^4}{4} \right) + 13500 \left( \frac{15 \times 7.5^2}{2} \right) \right]$$

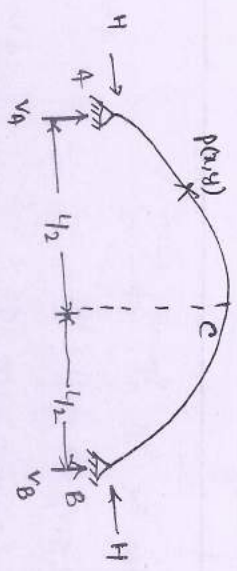
$$= 32167.964 + 25628.906 = 57796.87$$

$$\int y^2 dx = 57796.87$$

$$\int y^2 dx = \frac{8}{15} \times h^2 L = \frac{8}{15} \times 6^2 \times 30 = 576$$

$$\therefore H = \frac{\int M^2 y dx}{\int y^2 dx} = \frac{57796.87}{576} = 100.345 \text{ KN}$$

\* A 2-hinged parabolic arch of span L and rise h carries UDL of w/m run over the whole span. Assuming  $I = I_0$  Secx, find the reactions at the horizontal thrust developed.



$$V_A = V_B = \frac{wL}{2}$$

Beam moment at point  $P(x, y)$  is given by

$$M_x = w_1 \times x - w_2 \times x \times \frac{x}{2}$$

$$= \left( \frac{wL}{2} \right) x - \frac{w_2 x^2}{2}$$

$$= \frac{w_1 x}{2} (L - x)$$

$$y = \frac{4hx(L-x)}{L^2}$$

Substitute



Horizontal thrust

$$H = \frac{\int M^1 y \, dx}{\int y^2 \, dx}$$

$$= \frac{\int \frac{wLx}{2}(L-x) \times \frac{4hx(L-x)}{L^2} \, dx}{\int \frac{16h^2 x^2 (L-x)^2}{L^4} \, dx}$$

$$= \left( \frac{2wLh}{L^2} \right) \int x^2 (L-x)^2 \, dx$$

$$\frac{16h^2}{L^4} \int x^2 (L-x)^2 \, dx$$

$$= \frac{2wLh}{L^2} \times \frac{L^3}{8h^2}$$

$$= \frac{wL^2}{8h}$$

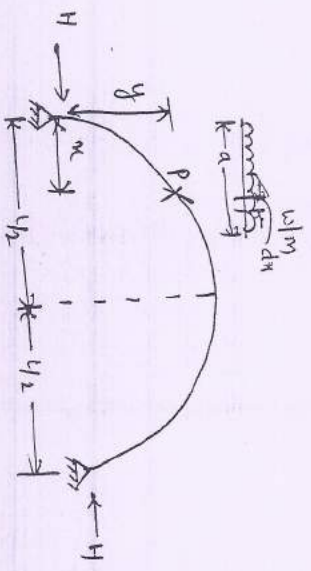
$$\therefore H = \frac{wL^2}{8h}$$

$\therefore$  If the load UDL is acted in half portion

$$\therefore H = \frac{1}{2} \times \frac{wL^2}{8h}$$

$$\therefore H = \frac{wL^2}{16h}$$

subjected to UDL of  $w$  per unit length over a length of  $l$  left support.



Consider the typical arch shown in figure. &

Let  $dx$  be an elemental length at  $P(x, y)$ .

Due to elemental load 'Mdx' acting on the element,

Horizontal thrust produced is given by

$$dH = \frac{5}{8} \left( \frac{w}{hl^3} \right) a(l-a)(l^2 + la - a^2)$$

$$= \frac{5}{8} \left( \frac{wM}{hl^3} \right) (l-x)(l^2 + lx - x^2)$$

$$= \frac{5}{8} \left( \frac{w}{hl^3} \right) (l^3x + lx^2 - x^3L - x^2l - lx^3 + x^4)$$

$$= \frac{5}{8} \left( \frac{w}{hl^3} \right) (l^3x - 2lx^3 + x^4) \, dx$$

$$H = \frac{5}{8} \left( \frac{w}{hl^3} \right) \int_0^l (l^3x - 2lx^3 + x^4) \, dx$$

$$= \frac{5}{8} \left( \frac{w}{hl^3} \right) \left[ \frac{l^3x^2}{2} - \frac{2lx^4}{4} + \frac{x^5}{5} \right]_0^l$$

$$= \frac{5}{8} \left( \frac{w}{hl^3} \right) \left[ \frac{l^3a^2}{2} - \frac{2la^4}{4} + \frac{a^5}{5} \right]$$

$$= \frac{2}{8} \left( \frac{w}{4L^3} \right) \left( \frac{1}{16} \right) (5L^2x^2 - 5La^2 + 2ax^3)$$

$$= \left( \frac{1}{16} \right) \left( \frac{w}{4L^3} \right) ax^2 (5L^2 - 5La^2 + 2ax^3)$$

\* A 2-hinged parabolic arch is loaded as shown in figure.

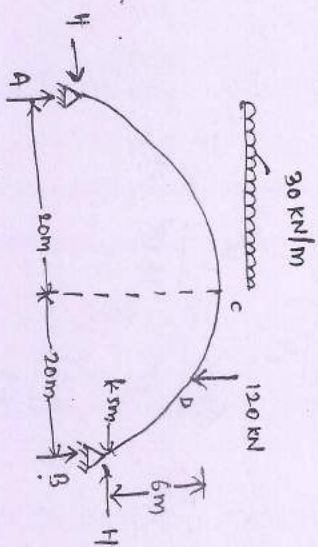
Determine the

i) horizontal thrust,

ii) max. +ve & -ve moments,

iii) Shear force & normal thrust at 10m from the left support

Assume  $\theta = \theta_0$  Sec  $\theta$ , where  $\theta_0$  is the moment of inertia at the crown and  $\theta$  is the slope at the section under consideration.



Let  $H_1$  be the horizontal thrust due to given UDL and  $H_2$  be due to given concentrated load

Then

$$H_1 = \frac{wL^2}{16h} = \frac{30 \times 40^2}{16 \times 6} = 500 \text{ kN}$$

$$H_2 = \frac{5}{8} \left( \frac{wL^2}{4L^3} \right) a(L-a) (L^2 + La - a^2)$$

$$= \frac{5}{8} \left( \frac{120}{6 \times 40^3} \right) 5(40-5) (40^2 + 40 \times 5 - 5^2)$$

$$H_2 = 60.669 \text{ kN}$$



$$H = M_1 + M_2$$

$$= (500 + 60 \cdot 669) \text{ kN}$$

$$H = 560.669 \text{ kN}$$

$$\sum V_A + V_B = 30 \times 20 + 120 \quad \text{--- (1)}$$

$$\sum M_B = 0, \quad V_A \times 40 - 30 \times 20 \left( \frac{20}{2} + 20 \right) - 120 \times 5 = 0$$

$$V_A = \frac{30 \times 20 \times 30 + 120 \times 5}{40}$$

$$V_A = 465 \text{ kN}$$

$$\therefore \text{From (1)}$$

$$V_B = 30 \times 20 + 120 - 465$$

$$\therefore V_B = 255 \text{ kN}$$

For the portion AC,

$$M_x = V_A \times x - 30 \times x \times \frac{x}{2} - H \times y$$

$$= 465x - 15x^2 - 560.669 \times \frac{(4)(6)x(40-x)}{40}$$

$$= 465x - 15x^2 - 8.41(40x - x^2)$$

$$= 128.6x - 6.59x^2$$

$$\therefore \text{For } M_{\text{max}}, \quad \frac{dM_x}{dx} = 0 = 128.6 - 13.18x$$

$$x = \frac{128.6}{13.18} = 9.757 \text{ m}$$

$$\therefore M_{\text{max}} = 128.6 \times 9.757 - 6.59 \times 9.757^2$$

$$= 128.6 \times 9.757 - 6.59 \times 9.757^2$$

$$= 627.388 \text{ kNm}$$

In portion CD, measuring x from right support,

$$M_x = V_B x - 120(x-5) - H y$$

$$= 255x - 120x + 600 - 8.41(40x - x^2)$$

$$= 600 - 201.4x + 8.41x^2$$

$$\therefore \frac{dM_x}{dx} = 0 = -201.4 + 16.82x$$

$$\therefore x = \frac{201.4}{16.82} = 11.974 \text{ m}$$

$$\therefore M_{\text{min}} = 600 - 201.4 \times 11.974 + 8.41 \times 11.974^2$$

$$= -605.766 \text{ kNm}$$

In portion BD,

$$M_x = V_B x - H y$$

$$= 255x - 8.41(40x - x^2)$$

$$= -8.41x + 8.41x^2$$

$$\therefore \frac{dM_x}{dx} = 0 = -8.41 + 16.82x$$

$$\therefore x = \frac{8.41}{16.82} = 0.5 \text{ m}$$

$$\therefore M = -8.41(4.839) + 8.41(4.839)^2$$

$$= -196.967 \text{ kNm}$$

and max.  $\rightarrow$  movement  $\approx 605$  mm at 11.944 m from the support

om from the right support,

$$\tan \theta = \frac{dy}{dx} = \frac{d}{dx} \left[ \frac{4hx(1-x)}{l^2} \right]$$

$$= \frac{4h}{l^2} (1-2x)$$

$$= \frac{4(6)}{40^2} (40-2(40))$$

$$= 0.3$$

$$\theta = 16.694^\circ$$

$$V = V_A - 120 = 255 - 120 = 135$$

$$N = V \sin \theta + H \cos \theta$$

$$= 135 \sin 16.694^\circ + 560.669 \cos 16.694^\circ$$

$$= 575.815 \text{ KN}$$

$$Q = V \cos \theta - H \sin \theta$$

$$= 135 \cos 16.694^\circ - 560.669 \sin 16.694^\circ$$

$$Q = -31.800 \text{ KN}$$

Determine H by carrying out suitable integrations for the above problem.

for the portion 0 to 20 m from end A

$$M^1 = \left( V_A x - \frac{30x^2}{2} \right) = 465x - \frac{30x^2}{2}$$

and from 5m to 20m from end B

$$M^1 = V_B x - 120(x-5)$$

$$= 255x - 120x + 600$$

$$= 135x + 600$$

$$\therefore \int M^1 V dx = \int_0^{20} (465x - 15x^2) \times \frac{(4)(6) \times x(40-x)}{40^2} dx$$

$$+ \int_0^5 255x \times \frac{(4)(6) \times x(40-x)}{40^2} dx$$

$$+ \int_0^{20} (135x + 600) \times \frac{(4)(6) \times x(40-x)}{40^2} dx$$



$$= \frac{24}{1600} \int_0^{20} (465x - 15x^2) (40x - x^2) dx$$

$$+ \frac{255 \times 24}{1600} \int_0^5 (40x^2 - x^3) dx$$

$$+ \frac{24}{1600} \int_0^{20} (135x + 600) (40x - x^2) dx$$

$$= \frac{24}{1600} \int_0^{20} 18600x - 1065x^2 + 15x^4 dx + \frac{(24)(255)}{1600} \int_0^5 (40x^2 - x^3) dx$$



$$+ \frac{1.4}{1000} \int_0^L (12000x + 48000 - 6000x^2) dx$$

$$= 249000 + 15888.344 + 175816.406$$

$$= 430593.75$$

$$\int y^2 dx = \frac{8}{15} \times k^2 L = \frac{8}{15} \times 6^2 \times 40 = 368$$

$$\therefore H = \frac{\int M^2 y^2 dy}{\int y^2 dx} = \frac{430593.75}{368}$$

$$H = 560.669 \text{ N}$$

\* EFFECT OF YIELDING OF SUPPORTS:-

$$H = \frac{\int M^2 y \left( \frac{ds}{EI} \right)}{\int y \frac{ds}{EI} + K}$$

K = 0.0001  
K - yielding of support for with half length

\* Effecting of shortening of end:-

$$H = \frac{\int M^2 y \frac{ds}{EI}}{\int y \left( \frac{ds}{EI} \right) + \frac{L}{EAM} + K}$$

Effect of Temperature Stress

$$H = \frac{\int M^2 y \left( \frac{ds}{EI} \right) + L \alpha t}{\int y \left( \frac{ds}{EI} \right) + \left( \frac{L}{EAM} \right) + K}$$

\* A two hinged parabolic arch of span 50m and rise 5 subjected to a central concentrated load of 60kN. It has an support which yields by 0.0001 mm/kN.

Taking  $E = 200 \text{ kN/mm}^2$ ,  $I = 5 \times 10^9 \text{ mm}^4$ ,

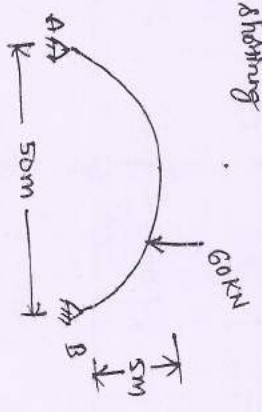
Average area  $A_m = 10000 \text{ mm}^2$ ,

$\alpha = 12 \times 10^{-6} / ^\circ\text{C}$

and assuming scant variation, calculate the horizontal developed when the temperature rises by  $20^\circ\text{C}$

(i) neglecting arch shortening.  
(ii) considering arch shortening.

(A)



taking left hinging A as the origin,

$$y = \frac{4h^2 x(50-x)^2}{L^2} = \frac{4 \times 5^2 \times x(50-x)^2}{50^2}$$

$$y = \frac{x(50-x)^2}{125}$$

$$y = \frac{50x-x^2}{125}$$

$$y'' dx = \int_0^{50} \frac{(50x - x^2)}{125} dx$$

$$= \frac{1}{125} \int_0^{50} (50x + x^2 - 100x^2) dx$$

$$= \frac{1}{125} \left[ 2500 \frac{x^2}{2} - 100 \frac{x^3}{3} + \frac{x^5}{5} \right]_0^{50}$$

$$= 666.67$$

Beam moment at A (M<sub>A</sub>) at x from A

$$M_1 = V_A \cdot x$$

$$= 30x$$

$$N = \int_0^{25} M_1 y dx = 2 \int_0^{25} 30x y dx$$

$$= 2 \int_0^{25} 30x \left( \frac{50x - x^2}{125} \right) dx$$

$$= 60 \int_0^{25} \frac{50x^2 - x^3}{125} dx$$

$$= \frac{60}{125} \left[ 50 \frac{x^3}{3} - \frac{x^4}{4} \right]_0^{25}$$

$$= \frac{48105}{EI}$$

$$\therefore V_A + V_B = 60$$

$$V_A = 30 = V_B$$

New,  $E = 200 \text{ kN/mm}^2$  and  $A_m = 1000 \text{ mm}^2$

converting E, I, A<sub>m</sub> to the metric unit

$$EI = 200 \times 10^6 \times 5 \times 10^9 \times 10^{-12}$$

$$= 1000000 \text{ KNm}^2$$

$$E A_m = 200 \times 10^6 \times 1000 \times 10^{-6}$$

$$= 2000000$$

$$\therefore \int M_1 y dx = 0.078125$$

New,  $L \Delta t = 50 \times 12 \times 10^6 \times 20 = 0.012$

yielding August per unit thickness

$$k = 0.0001$$

$\therefore$  neglecting rib shortening,

$$A = \frac{\int M_1 y \left( \frac{dx}{EI} \right) + \Delta t \cdot k}{\int y^2 \left( \frac{dx}{EI} \right) + 0.012}$$

$$= \frac{0.078125 + 0.012}{\frac{(666.67)}{1000000} + 0.0001}$$

$$= \frac{0.090125}{7.669 \times 10^{-4}}$$

$$= 119.554 \text{ kN}$$

$\therefore \frac{L}{E A_m}$  : rib shortening

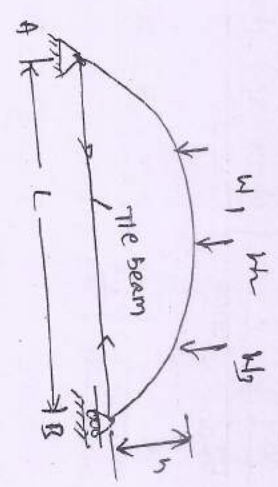


Rib shortening considered,

$$\begin{aligned}
 \Delta &= \int \frac{y^2 ds}{EI} + \frac{L}{EA} + k \\
 &= \frac{0.7854 \times 25^3 + 0.012}{7.66 \times 10^4} + \frac{50}{1000000} \\
 &= 113.841 \mu\text{m}
 \end{aligned}$$

Tied Arch:-

$A_T$  - Average tie beam  
 $E_T$  - Modulus of elasticity



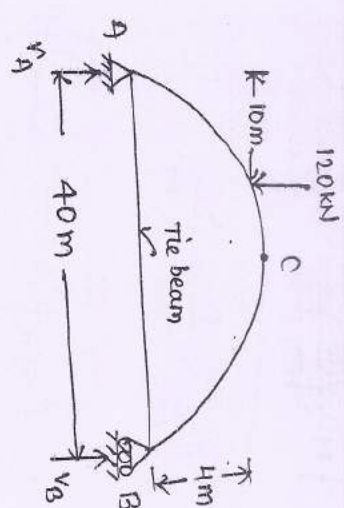
$$\therefore K = \frac{L}{A_T E_T}$$

$$\begin{aligned}
 \therefore H &= \frac{\int M^2 y \frac{ds}{EI} + L \times T}{\int y^2 \frac{ds}{EI} + \frac{L}{EA} + \frac{L}{A_T E_T}}
 \end{aligned}$$

shown in figure. Determine the axial force developed in the tie beam and moment at the Crown in arch. Consider ribs shortening. Assume slant moment of inertia of the arch ribs.

Given  
 Spigate area of ribs =  $18000 \text{ mm}^2$  (A)  
 $E_T$  of ribs =  $200 \text{ kN/mm}^2$  ( $E_T$ )  
 $EI$  of ribs =  $150000 \text{ kNm}^2$

Curved length of tie =  $4000 \text{ mm}$  (Average axial)  
 $E$  of tie =  $2000 \text{ kN/mm}^2$



$$\begin{aligned}
 \sum F_V &= 0, & V_A + V_B &= 120 \\
 \sum M_A &= 0, & V_B \times 40 &= 120 \times 10
 \end{aligned}$$

$$\begin{aligned}
 &V_B = 30 \text{ kN} \\
 &V_A = 90 \text{ kN}
 \end{aligned}$$

the value of the displacement is 57 mm. Now by using  $\alpha$  from A:

$$\Delta = V_A \alpha = 90 \alpha, \text{ when } 0 < \alpha < 10 \text{ m}$$

$$= V_B (L - \alpha) = 30(40 - \alpha), \text{ when } 10 < \alpha < 40$$

the  $\alpha$  varies in span from

$$\int \frac{dS}{EI} = \int \frac{dx}{EI}$$

Considering rib shortening of the tied arch

$$H = \frac{\int m^1 y \left( \frac{dS}{EI_0} \right)}{\int y^2 \frac{dS}{EI} + \frac{L}{EA_m} + \frac{L}{A_T E_T}}$$

$$\int H^1 y \frac{dS}{EI_0} = \frac{1}{EI_0} \int_0^{10} 90 \alpha \left[ \frac{4 \times 4 \times \alpha \times (40 - \alpha)}{40^2} \right] dx$$

$$+ \frac{1}{EI_0} \int_{10}^{30} 30(40 - \alpha) \left[ \frac{4(4) \alpha (40 - \alpha)}{40^2} \right] dx$$

$$= \frac{0.9}{EI_0} \int_0^{10} (40\alpha^2 - \alpha^3) dx + \frac{0.3}{EI_0} \int_{10}^{30} \alpha (40 - \alpha)^2 dx$$

$$= \frac{0.9}{EI_0} \left[ \frac{40 \alpha^3}{3} - \frac{\alpha^4}{4} \right]_0^{10} + \frac{0.3}{EI_0} \left[ \int_0^{10} \alpha (1600 + 80\alpha - 30\alpha^2) \right]$$

$$= \frac{0.9}{EI_0} \left[ \frac{40000}{3} - \frac{10000}{4} \right] + \frac{0.3}{EI_0} \left[ \frac{1600 \alpha^2}{2} + \frac{80 \alpha^3}{3} - \frac{80 \alpha^3}{3} \right]_0^{30}$$

$$= \frac{57000}{EI_0} = \frac{57000}{150000} = 0.38$$

$$\therefore D = \int_0^{40} y^2 \frac{dx}{EI_0} = 2 \int_0^{20} \left( \frac{4(4) \alpha (40 - \alpha)}{40} \right)^2 \frac{dx}{EI_0}$$

$$= \frac{2}{EI_0} \times \frac{1}{10000} \int_0^{20} (40\alpha - \alpha^2)^2 dx$$

$$= \frac{2}{10000 EI_0} \int_0^{20} (1600\alpha^2 + \alpha^4 - 80\alpha^3) dx$$

$$= \frac{2}{10000 EI_0} \left[ \frac{1600 \alpha^3}{3} + \frac{\alpha^5}{5} - \frac{80 \alpha^4}{4} \right]_0^{20}$$

$$= \frac{341.333}{EI_0} = 2.27555 \times 10^{-3}$$

$\frac{40 \times 40}{1600}$

$\therefore H =$

$$\frac{0.38}{2.24555 \times 10^{-3} + \left( \frac{40}{200 \times 10^6 \times 18000 \times 10} \right) + \left( \frac{40}{200 \times 10^6 \times 40000 \times 10} \right)}$$

$$= 162.625 \text{ KN}$$

$\therefore$  Maximum  $= V_B \times 20 - H \times 4$

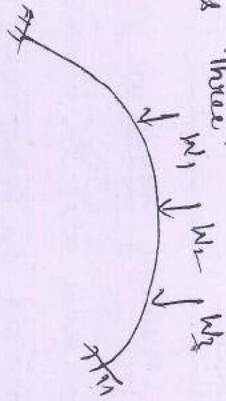
$$= 30 \times 20 - 162.625 \times 4 = -50.5 \text{ kNm}$$



## FIXED ARCH

Fixed arches are also called hingeless arches.

At each end there are 3 reaction components, thus giving a total of six reaction components. Only three independent static equilibrium equations are available. Hence, degree of indeterminacy is three.



To analyse such arches

two methods are available

1. Conjoint deformation method
2. Elastic centre method.

LATERAL LOAD ANALYSIS USING APPROXIMATE METHODS :-

\* Approximate methods for building frames

1. Portal method
2. Cantilever method

Multistorey frames are subjected to horizontal

forces due to wind and seismic forces. The horizontal forces are assumed to act at joints and any one method may be used to their analysis.

1. Portal method :-

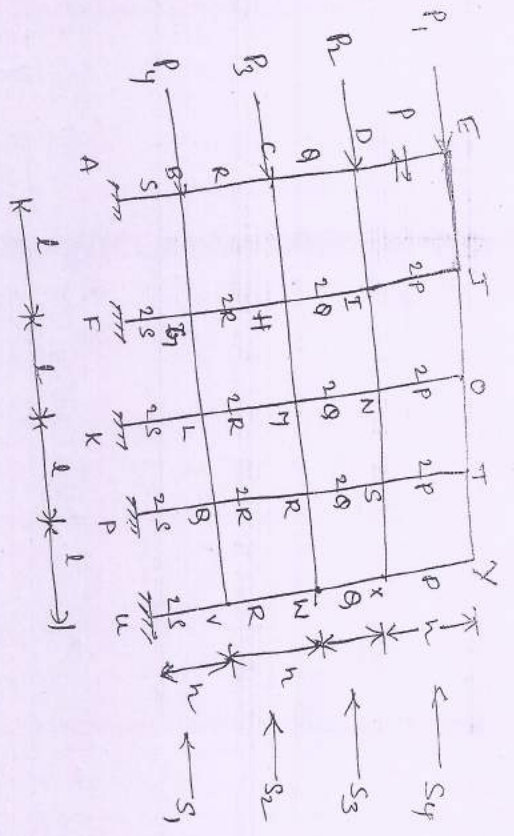
In this method, the following two assumptions are made.

- i. Point of contraflexure occurs at the middle of all members of the frame
- ii Horizontal shear taken by each interior column is double of that taken by external columns.

• With the above two assumptions, the frame becomes a determinate and hence analysis is easy.

This method was presented by Albert Smith in the 'Journal of Western Society of Engineers' in 1915.





1) Column Shear

consider column S<sub>4</sub>:

$$P + 2P + 2P + 2P + P = P_1$$

$$\therefore 8P = P_1$$

$$\therefore P = \frac{P_1}{8}$$

$\therefore$  Column shear in outer column =  $\frac{P_1}{8}$   
 " in inner " =  $2P = \frac{2P_1}{8} = \frac{P_1}{4}$

Column S<sub>3</sub>:-

$$8Q = P_1 + P_2$$

$$Q = \frac{P_1 + P_2}{8}$$

S.F in outer,  $Q = \frac{P_1 + P_2}{8}$

S.F in inner,  $2Q = \frac{P_1 + P_2}{4}$

$$8R = P_1 + P_2 + P_3$$

$$\therefore R = \frac{P_1 + P_2 + P_3}{8}$$

S.F in outer column  $R = \frac{P_1 + P_2 + P_3}{8}$

" in inner, "  $2R = \frac{P_1 + P_2 + P_3}{4}$

Column S<sub>1</sub>:-

$$8S = P_1 + P_2 + P_3 + P_4$$

$$S = \frac{P_1 + P_2 + P_3 + P_4}{8}$$

S.F in outer column  $S = \frac{P_1 + P_2 + P_3 + P_4}{8}$

" in inner "  $2S = \frac{P_1 + P_2 + P_3 + P_4}{4}$

2) Column moments:-

Column Moment = S.F  $\times$  half (height of T)

3) Beam Moment:-

At point E,

$$M_{ED} = M_{ET}$$

At J:-

$$M_{JE} + M_{JT} = M_{JI}$$

$$\therefore M_{JT} = M_{JI} - M_{JE}$$

At point O:-

$$M_{OJ} + M_{OT} =$$

$$\therefore M_{OT} = M_{ON}$$

At point T:-

$$M_{OT} + M_{TY}$$

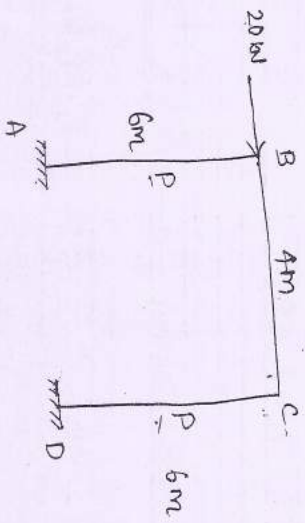
$$\therefore M_{TY} = M_T$$

$$B.M = S.F \times \frac{L}{2}$$

$$\therefore S.F = B.M \times \frac{2}{L}$$

Use the portal frame shown in the figure by

method



Let 'P' be the Shear force in the column

Column Shear :-

$$\therefore P + P = 20$$

$$2P = 20$$

$$P = 10 \text{ kN}$$

$$\therefore \text{S.F in column } AB = 10 \text{ kN}$$

$$\text{" " " } CD = 10 \text{ kN}$$

Column Moment :-

$$\text{Column moment} = S.F \times \frac{h}{2}$$

$$\therefore M_{AB} = 10 \times \frac{6}{2} = 30 \text{ kN-m}$$

$$\therefore M_{CD} = 10 \times \frac{6}{2} = 30 \text{ kN-m}$$

iii) Beam Moment :-

At point B:

$$M_{BC} = M_{AB} = 30 \text{ kN-m}$$

iv) Beam Shear :-

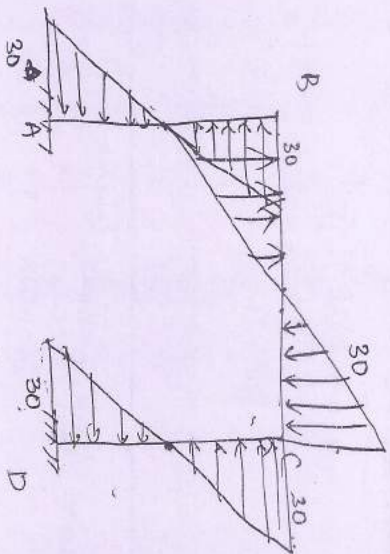
$$B.M = S.F \cdot B \times \frac{L}{2}$$

$$\therefore S.F \cdot B = \frac{B.M}{L}$$

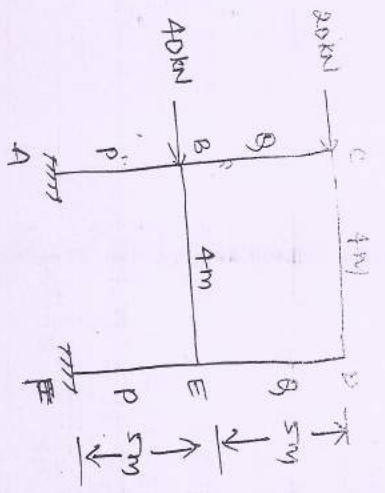
$$M_{BC} = S.F \text{ in beam} \times \frac{4}{2}$$

$$30 = S.F \cdot B \times 2$$

$$S.F \cdot B = 15 \text{ kN}$$







Column Shear:

Let 'Q' be the S.F in column CB & DE

$$Q + Q = 20$$

$$2Q = 20$$

$$Q = 10 \text{ kN}$$

$\therefore$  S.F in column CB & DE = 10 kN //

Let 'P' be the S.F in column AB & FE

$$P + P = 20 + 40$$

$$\therefore 2P = 60$$

$$P = 30 \text{ kN}$$

$\therefore$  S.F in column AB & FE = 30 kN //

ii) Column moments:-

Column moment = S.F  $\cdot$  C  $\times$   $\frac{h}{2}$

$$\therefore M_{CB} = M_{DE} = Q \times \frac{5}{2} = 10 \times \frac{5}{2} = 25 \text{ kN-m}$$

$$\therefore M_{AB} = M_{FE} = P \times \frac{5}{2} = 30 \times \frac{5}{2} = 75 \text{ kN-m}$$

iv) Beam Shear:-

At joint C:-

$$M_{CB} = M_{CD} = 25 \text{ kN-m}$$

At joint B:-

$$M_{CB} + M_{B4} = M_{BE}$$

$$\therefore M_{BE} = 25 + 75 = 100 \text{ kN-m}$$

iv) Beam Shear:-

$$M_{CD} = S.F \times \frac{L}{2}$$

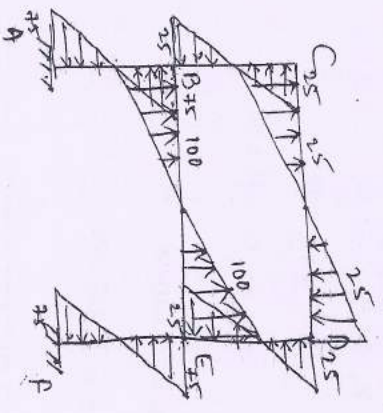
$$25 = S.F \text{ of beam } CD \times \frac{4}{2}$$

$$\therefore \text{S.F on beam } CD = 25 \times \frac{2}{4} = 12.5 \text{ kN} //$$

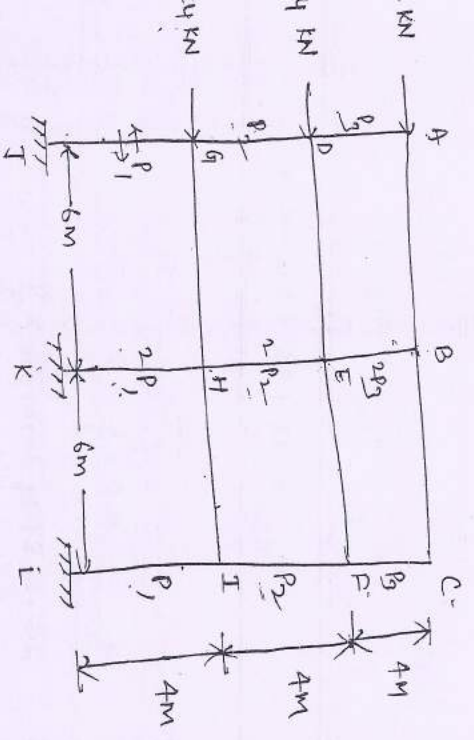
$$M_{BE} = S.F \text{ on beam } BE \times \frac{L}{2}$$

$$100 = S.F \cdot BE \times \frac{4}{2}$$

$$\therefore \text{S.F of beam } BE = 50 \text{ kN} //$$



Wind loads, wind loads transferred to joints A, D & G  
 2kN, 24kN and 24kN respectively. Analyse the frame by  
 method.



Let the horizontal shears taken by each column be

to existing columns  $P_1, P_2, P_3$  and for

for interior columns be  $2P_1, 2P_2, 2P_3$

∴ Let  $P_3$  be the s.f in columns AD & CF and  $2P_3$  be the BE

$$P_3 + 2P_3 + P_3 = 12$$

$$4P_3 = 12$$

$$P_3 = 3 \text{ kN}$$

∴ s.f in column AD & CF =  $3 \text{ kN}$

& s.f in column BE =  $6 \text{ kN}$

iii)  $P_2 + 2P_2 + P_2 = 12 + 24$

$$4P_2 = 36$$

∴ s.f in column DG & FI =  $9 \text{ kN}$

$$EH = 18 \text{ kN}$$

$$P_2 = 9 \text{ kN}$$

iii)

$$P_1 + 2P_1 + P_1 = 12 + 24 + 24$$

$$4P_1 = 60$$

$$P_1 = 15 \text{ kN}$$

∴ s.f in column GJ & IL =  $15 \text{ kN}$

$$HK = 30 \text{ kN}$$

ii) Column moment:-

= s.f in column  $\times \frac{h}{2}$

$$M_{AD} = M_{CF} = 3 \times \frac{4}{2} = 6 \text{ kN-m}$$

$$M_{BE} = 6 \times \frac{4}{2} = 12 \text{ kN-m}$$

$$M_{DG} = M_{FI} = 9 \times \frac{4}{2} = 18 \text{ kN-m}$$

$$M_{EH} = 18 \times \frac{4}{2} = 36 \text{ kN-m}$$

$$M_{GJ} = M_{IL} = 15 \times \frac{4}{2} = 30 \text{ kN-m}$$

$$M_{HK} = 30 \times \frac{4}{2} = 60 \text{ kN-m}$$

iii) Beam moment:-

At point A:-

$$M_{AD} = M_{AB} = 6 \text{ kN-m}$$

At point B:-

$$M_{AB} + M_{BC} = M_{BE}$$

$$\therefore M_{BC} = M_{BE} - M_{AB} = 12 - 6 = 6 \text{ kN-m}$$



At point D:

$$M_{AD} + M_{DG} = M_{DE}$$

$$6 + 18 = M_{DE}$$

$$\therefore M_{DE} = 24 \text{ KN-m} //$$

At point F:-

$$\therefore M_{EF} = M_{CF} + M_{FI}$$

$$= 6 + 18 = 24 \text{ KN-m} //$$

$$\therefore M_{EF} = 24 \text{ KN-m} //$$

At point G:-

$$\therefore M_{GH} = M_{DG} + M_{GT} = 18 + 30 = 48 \text{ KN-m} //$$

At point I:-

$$\therefore M_{HI} = M_{FI} + M_{IL} = 18 + 30 = 48 \text{ KN-m} //$$

(iv) Beam Shear:-

S.F on AB:-

$$\therefore M_{AB} = \text{s.f} \times \frac{L}{2}$$

$$6 = \text{s.f on AB} \times \frac{6}{2}$$

$$\therefore \text{s.f on AB} = 2 \text{ KN} = \text{s.f on BC}$$

S.F on DE&EF:-

$$\therefore M_{DE} = \text{s.f on DE} \times \frac{L}{2}$$

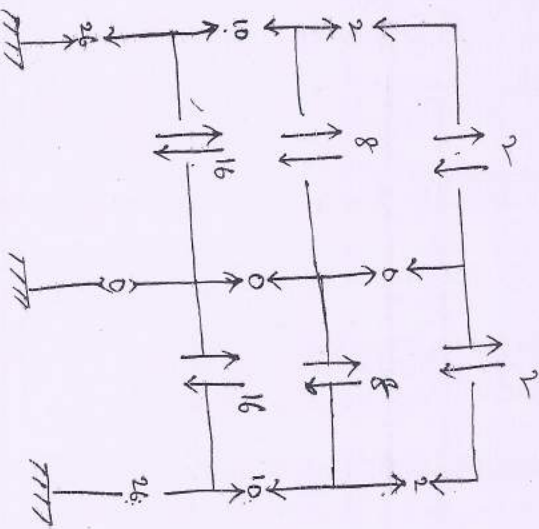
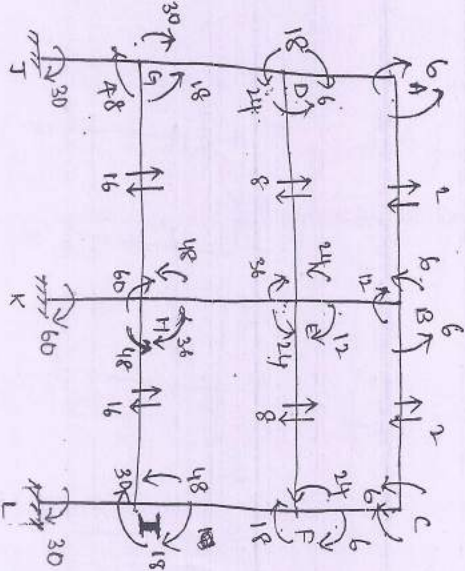
$$24 = \text{s.f on DE} \times \frac{6}{2}$$

$$\therefore \text{s.f on DE} = 8 \text{ KN} = \text{s.f on EF}$$

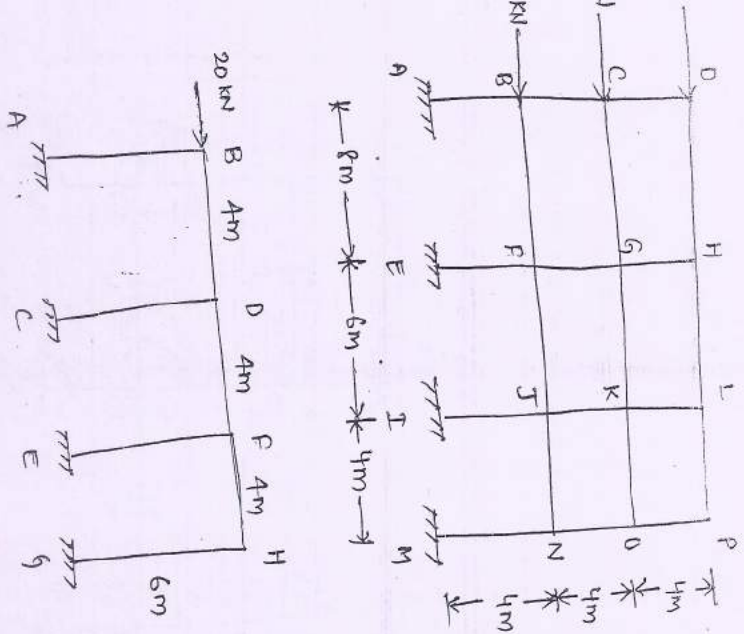
$$\therefore M_{GH} = \text{s.f on GH} \times \frac{L}{2}$$

$$48 = \text{s.f} \times \frac{6}{2}$$

$$\therefore \text{s.f on GH} = 16 = \text{s.f on HI} //$$



Vertical loads



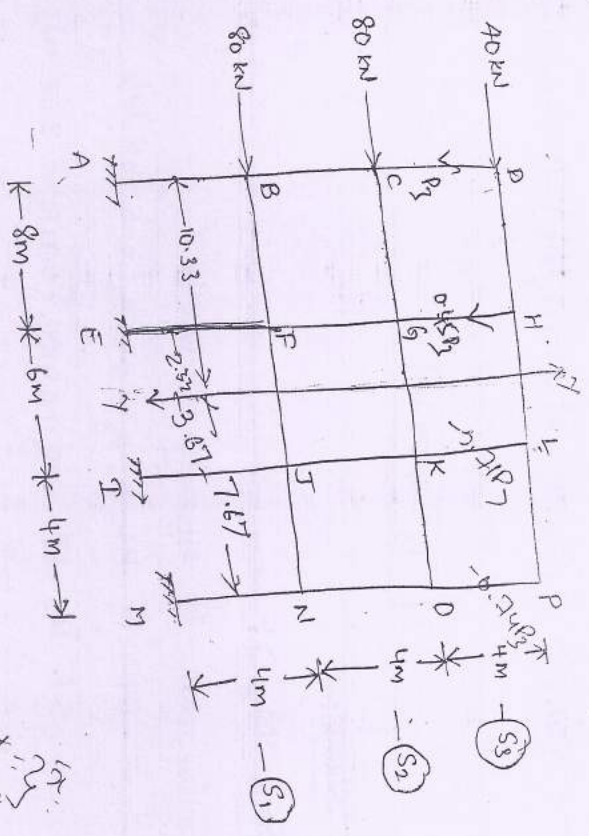
2. CANTILEVER METHOD :

The following assumptions are made in this method

- There is a point of contraflexure at the center of each member
- The intensity of axial stress in each column of a story is proportional to the horizontal distance of that column from the centre of gravity of all the columns of the story under consideration (The analysis is carried out story by story)
- The area of each intermediate column is twice that of area of exterior of end column.



Analysis of frame system below by column method



Consider above assumptions

Let the area of each column = A

and " " intermediate " = 2A

∴ Total area of column = 6A

Let the C.G. of the column be  $\bar{x}$  from the axis of column ABCD

$$\bar{x} = \frac{A_1 x_1 + A_2 x_2 + \dots + A_n x_n}{A_1 + A_2 + \dots + A_n}$$

$$\bar{x} = \frac{(A \times 0) + (2A \times 8) + (2A \times 14) + (A \times 18)}{6A}$$

$$\bar{x} = \frac{16A + 28A + 18A}{6A} = \frac{62A}{6A}$$

$$\bar{x} = 10.33 \text{ m}$$

Let the axial force in the column AB =  $P_1$

Stress in the column AB =  $\frac{\text{distance} \times P_1}{A} = \frac{10.33 \times P_1}{10.33 \times A} = \frac{P_1}{A}$

Stress in the column EF =  $\frac{2.33}{10.33} \times \frac{P_1}{A} = 0.225 \frac{P_1}{A}$

Stress in the column IJ =  $\frac{-3.67}{10.33} \times \frac{P_1}{A} = -0.355 \frac{P_1}{A}$

Stress in the column MN =  $\frac{-7.67}{10.33} \times \frac{P_1}{A} = -0.74 \frac{P_1}{A}$

∴ Axial force in Column AB = Stress in column  $\times$  Area

$$= \frac{P_1}{A} \times A = P_1 \quad (\text{Tension})$$

$$\text{EF} = 0.225 \times \frac{P_1}{A} \times 2A = 0.45 P_1$$

$$\text{IJ} = 0.355 \times \frac{P_1}{A} \times 2A = 0.71 P_1$$

$$\text{MN} = 0.74 \times \frac{P_1}{A} \times A = 0.74 P_1$$

New taking moments about the point of contraflexure of column AB

$$40 \times (4 + 4 + 2) + 80 \times 2 + P_1 \times 0 + 0.45 P_1$$

$$- 0.71 P_1 \times 14 - 0.74 P_1 \times 18 = 0$$

$$400 + 480 + 160 + 3.6 P_1 - 9.8 P_1 - 13.32 P_1 = 0$$

$$19.52 P_1 = 1040$$

42 :-

Let  $P_2$  be the axial force in the column BC

at joint in the column BC =  $P_2$  (T)

" " "  $F_9 = \frac{2.33}{10.33} \times \frac{P_2}{A} \times 12A = 0.45 P_2$  (T)

" " " JK =  $0.71 P_2$  (C)

" " " NO =  $0.74 P_2$  (C)

two taking moments about point of centre of mass of BC.

$40 \times (4+2) + 80 (2) + P_2 \times 0 + (0.45 P_2 \times 8)$

$- (0.71 P_2 \times 14) - (0.74 P_2 \times 18) = 0$

$400 = 19.52 P_2$

$\therefore P_2 = 20.49 \text{ kN}$

Step 3: Let ' $P_3$ ' be the axial force in the column 'CD'

axial force in column CD =  $P_3$  (T)

" " " GH =  $0.45 P_3$  (T)

" " " KL =  $0.71 P_3$  (C)

" " " OP =  $0.74 P_3$  (C)

two taking moments about point of centre of mass of BC

$(40 \times 2) + (0.45 P_3 \times 8) - (0.71 P_3 \times 14) - (0.74 P_3 \times 18) = 0$

$80 = 19.52 P_3$

$\therefore P_3 = 4.09 \text{ kN}$

BEAM SHEAR :-

From step 3 :-

S-F in DH =  $P_3 = 4.10 \text{ kN}$

S-F in HL =  $P_3 + 0.45 P_3 = 5.945 \text{ kN}$

S-F in LP =  $P_3 + 0.45 P_3 - 0.71 P_3 = 3.07 \text{ kN}$

(8)  $0.74 P_3 = 3.03 \text{ kN}$

From step 2 :-

S-F in CG =  $P_2 - P_3 = 16.36 \text{ kN}$

S-F in GK =  $P_2 + 0.45 P_2 - P_3 - 0.45 P_3 = 23.76 \text{ kN}$

S-F in KO =  $P_2 + 0.45 P_2 - 0.71 P_2 - P_3 - 0.45 P_3 + 0.7 P_3$

$= 12.29 \text{ kN}$

Step 1 :-

S-F in BF =  $P_1 - P_2 = 32.79 \text{ kN}$

S-F in FJ =  $P_1 + 0.45 P_1 - P_2 - 0.45 P_2 = 47.54 \text{ kN}$

S-F in JN =  $P_1 + 0.45 P_1 - 0.7 P_1 - P_2 - 0.45 P_2 + 0.7 P_2 = 24.59 \text{ kN}$

Beam Moment :-

Beam moment = shear force in beam x half span



$$M_{DH} = 4.1 \times 4 = 16.4 \text{ kN-m}$$

$$M_{HL} = 5.945 \times 3 = 17.835 \text{ kN-m}$$

$$M_{LP} = 3.04 \times 2 = 6.14 \text{ kN-m}$$

$$M_{CG} = 16.36 \times 4 = 65.44 \text{ kN-m}$$

$$M_{GH} = 23.26 \times 3 = 69.78 \text{ kN-m}$$

$$M_{KD} = 12.29 \times 2 = 24.58 \text{ kN-m}$$

$$M_{BF} = 32.7 \times 4 = 130.8 \text{ kN-m}$$

$$M_{FT} = 47.54 \times 3 = 142.62 \text{ kN-m}$$

$$M_{TN} = 24.59 \times 2 = 49.18 \text{ kN-m}$$

column moment :-

@ point D  
 $M_{DC} = M_{DH} = 16.4 \text{ kN-m}$

@ point H  
 $M_{HG} = M_{DH} + M_{HL} = 16.4 + 17.835 = 34.235 \text{ kN-m}$

@ point L  
 $M_{LK} = M_{HL} + M_{LP} = 17.835 + 6.14 = 23.97 \text{ kN-m}$

@ point P  
 $M_{PD} = M_{LP} = 6.14 \text{ kN-m}$

@ point C  
 $M_{CG} = M_{DC} + M_{CB}$   
 $\therefore M_{CB} = M_{CG} - M_{DC} = 65.44 - 16.4 = 49.04 \text{ kN-m}$

@ point G

$$M_{GC} + M_{gk} = M_{gH} + M_{gF}$$

$$M_{gF} = M_{gC} + M_{gk} - M_{gH}$$

$$= 65.44 + 69.78 - 34.235 = 100.98 \text{ kN-m}$$

@ point K

$$M_{KQ} + M_{KD} = M_{KL} + M_{KT}$$

$$69.78 + 24.58 = 23.99 - M_{KT}$$

$$\therefore M_{KT} = 70.39 \text{ kN-m}$$

@ point D

$$M_{PD} + M_{DN} = M_{DK} \Rightarrow M_{DN} = M_{DK} - M_{PD}$$

$$M_{DN} = 18.44 \text{ kN-m}$$

@ point B

$$M_{BA} + M_{BC} = M_{BF}$$

$$\therefore M_{BA} = 81.76 \text{ kN-m}$$

@ point F

$$M_{FA} + M_{FT} = M_{fg} + M_{fE}$$

$$M_{fE} = 172.44 \text{ kN-m}$$

@ point J

$$M_{JF} + M_{JN} = M_{JK} + M_{JT}$$

$$M_{JT} = 121.41 \text{ kN-m}$$

@ point N

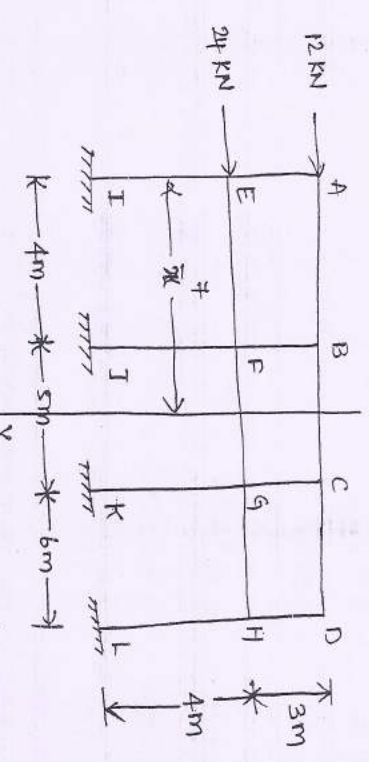
$$M_{ND} + M_{NM} = M_{TN}$$

$$M_{NM} = 30.74 \text{ kN-m}$$

Column moment = S.F x half distance

- S.F DC =  $M_{DC} \times \frac{2}{5}$   
 $= 16.4 \times \frac{2}{5}$   
 $= 16.4/2 = 8.2 \text{ KN}$
- S.F HQ =  $M_{HQ} / L = 34.235 / 2 = 17.117 \text{ KN}$
- S.F LK =  $\frac{28.97}{2} = 11.98 \text{ KN}$
- S.F PD =  $\frac{6.14}{2} = 3.07 \text{ KN}$
- S.F CB =  $\frac{49.04}{2} = 24.52 \text{ KN}$
- S.F GF =  $\frac{100.98}{2} = 50.49 \text{ KN}$
- S.F KJ =  $\frac{70.39}{2} = 35.19 \text{ KN}$
- S.F ON =  $\frac{18.44}{2} = 9.22 \text{ KN}$
- S.F BA =  $\frac{81.76}{2} = 40.88 \text{ KN}$
- S.F FE =  $\frac{172.44}{2} = 86.22 \text{ KN}$
- S.F JI =  $\frac{121.41}{2} = 60.7 \text{ KN}$
- S.F NM =  $\frac{30.74}{2} = 15.37 \text{ KN}$

\* Analyse the frame shown in figure by cantilever method. Take area-retained area of all columns as the same.



sol: considering the assumptions in cantilever method  
 let the area of each column be 'A' given 'A' same

Total area = 4A

$$\therefore \bar{x} = \frac{(A \times 0) + (A \times 4) + (A \times 9) + (A \times 15)}{4A}$$

$$\bar{x} = \frac{28A}{4A} = 7$$

From Story ① let the axial force in the column be  $P_1$

axial force in column EI = stress in the column EI x Area

$$= \frac{\text{distance} \times P_1}{\bar{x}} \times A$$

$$= \frac{7}{7} \times \frac{P_1}{A} \times A = P_1 \quad (T)$$

" in FJ =  $\frac{3}{7} \times \frac{P_1}{A} \times A = 0.428 P_1 \quad (T)$

" in GK =  $\frac{2}{7} \times \frac{P_1}{A} \times A = 0.285 P_1 \quad (C)$

" in HL =  $\frac{8}{7} \times \frac{P_1}{A} \times A = 1.142 P_1 \quad (C)$





Now taking moments about the point of contact of column  
EI

$$12 \times (3+2) + 24 \times 2 + P_1 \times 0 + (0.428 P_1 \times 4) - (0.285 P_1 \times 9) - (1.142 P_1 \times 15) = 0$$

$$108 = -1.712 P_1 + 2.565 P_1 + 17.13 P_1$$

$$108 = 17.983 P_1$$



Let  $P_2$  be the axial force in AE

From story ②

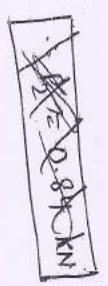
- AE =  $P_2$  (T)
- BF =  $0.428 P_2$  (T)
- CG =  $0.285 P_2$  (C)
- DH =  $1.142 P_2$  (C)

Now taking moments about the point of contact of column

$$12 \times 1.5 + P_2 \times 0 + (0.428 P_2 \times 4) - (0.285 P_2 \times 9) - (1.142 P_2 \times 15) = 0$$

$$18 = 17.983 P_2$$

$$P_2 = 1.0 \text{ kN}$$



BEAM SHEAR :-

From story ①

S.F in AB =  $P_2 = 1 \text{ kN}$

S.F in BC =  $P_2 + 0.428 P_2 = 1 + 0.428 = 1.428$

S.F in CD =  $P_2 + 0.428 P_2 - 0.285 P_2 = 1.428 - 0.285 = 1.143 \text{ kN}$

From story ②

S.F in EF =  $P_1 - P_2 = 6 - 1 = 5 \text{ kN}$

S.F in FG =  $P_1 + 0.428 P_1 - P_2 - 0.428 P_2 = 6 + (0.428 \times 6) - 1 - 0.428 = 7.14 \text{ kN}$

S.F in GH =  $P_1 + 0.285 P_1 - P_2 - 0.428 P_2 + 0 = 6 + 0.285 \times 6 - 1 - 0.428 \times 1 + 0.2 = 6.567 \text{ kN}$

BEAM MOMENT :-

Beam moment = S.F x half span

$M_{AB} = 1 \times \frac{4}{2} = 2 \text{ kNm}$

$M_{BC} = 1.428 \times \frac{5}{2} = 3.57 \text{ kNm}$

$M_{CD} = 1.143 \times \frac{6}{2} = 3.429 \text{ kNm}$

$M_{EF} = 5 \times \frac{4}{2} = 10 \text{ kNm}$

$M_{FG} = 7.14 \times \frac{5}{2} = 17.85 \text{ kNm}$

$M_{GH} = 6.567 \times \frac{6}{2} = 19.70 \text{ kNm}$

results:-

@ point A

$$M_{AE} = M_{AB} = 2 \text{ kNm}$$

@ point B

$$M_{BF} = M_{AB} + M_{BC} \\ = 2 + 3.57 = 5.57 \text{ kNm}$$

@ point C

$$M_{Cg} = M_{BC} + M_{CD} \\ = 3.57 + 3.429 = 6.99 \text{ kNm}$$

@ point D

$$M_{DH} = M_{CD} = 3.429 \text{ kNm}$$

@ point E

$$M_{EI} + M_{AE} = M_{EF} \\ \therefore M_{EI} = M_{EF} - M_{AE} = 10 - 2 = 8 \text{ kNm}$$

@ point F

$$M_{EF} + M_{Fg} = M_{BF} + M_{FJ} \\ M_{FJ} = M_{EF} + M_{Fg} - M_{BF} \\ = 10 + 19.85 - 5.57 = 24.28 \text{ kNm} \\ M_{FJ} = 22.28 \text{ kNm}$$

@ point G

$$M_{Fg} + M_{gH} = M_{Cg} + M_{gK} \\ M_{gK} = M_{Fg} + M_{gH} - M_{Cg} \\ = 24.28 + 19.70 - 6.99 = 37.0$$

@ point H

$$M_{gH} = M_{DH} + M_{HgL} \\ \therefore M_{HL} = M_{gH} - M_{DH} \\ = 19.70 - 3.429 \\ M_{HL} = 16.271 \text{ kNm}$$

Column shear:-

column moment =  $S.F \times \frac{\text{height of column}}{2}$

$\therefore S.F \text{ in } AE = \text{Column moment } M \times \frac{2}{h}$

- S.F in AE =  $2 \times \frac{2}{3} = 1.33 \text{ kN}$
- S.F in BF =  $5.57 \times \frac{2}{3} = 3.71 \text{ kN}$
- S.F in CG =  $6.99 \times \frac{2}{3} = 4.66 \text{ kN}$
- S.F in DH =  $3.429 \times \frac{2}{3} = 2.28 \text{ kN}$
- S.F in EI =  $8 \times \frac{2}{4} = 4 \text{ kN}$
- S.F in FJ =  $\frac{22.28 \times 2}{2} = 11.14 \text{ kN}$
- S.F in gK =  $\frac{30.56}{2} = 15.28 \text{ kN}$
- S.F in HL =  $\frac{16.271}{2} = 8.13 \text{ kN}$